IA-III - DTS – Answer Key

PART $A - 5 \times 2 = 10$ Marks

1. Define Progression Ratio

Progression Ratio (R) = $(Maximum\ speed\ /\ Minimum\ speed)^(1/(n-1))$ Where n is the number of speeds. It determines the geometric progression in speed steps in a gearbox.

2. Select the spindles for the 12-speed between 50 rpm to 600 rpm

Use geometric progression:

$$R=\left(\frac{600}{50}\right)^{1/11}\approx 1.174$$

- Speeds: 50, 58.7, 68.9, 80.9, 95.1, 111.8, 131.5, 154.7, 181.9, 213.9, 251.4, 296.3, ~600 rpm
- Use 3-shaft or 4-shaft arrangement depending on space and performance.

3. List out the methods for changing speeds in gear box

- Sliding mesh
- o Constant mesh
- o Synchromesh
- o Epicyclic gear trains
- o Multistage gear systems

4. Compare brake and clutch

Feature	Brake	Clutch
Function	Stops motion	Transmits motion
Location	On driven shaft	Between engine & transmission
Operated	Often manually or hydraulically	Automatically or manually
Energy	Absorbs	Transmits

5. List out the functions of a clutch

- Connects and disconnects engine power
- o Enables smooth starting
- Allows gear shifting
- o Provides overload protection

PART B $- 2 \times 13 + 1 \times 14 = 40$ **Marks**

6. (a) Nine-speed gear box (180 - 1800 rpm)

$$R = \left(rac{1800}{180}
ight)^{1/8} pprox 1.333$$

Speeds: 180, 240, 320, 427, 570, 760, 1013, 1350, 1800 rpm

Diagram:

Speed diagram: 3x3 arrangement

· Kinematic layout: 3 shafts with cone pulley arrangement

OR

(b) Six-speed gear box (100 - 1000 rpm)

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$$R = \left(\frac{1000}{100}\right)^{1/5} pprox 1.585$$

Speeds: 100, 158, 251, 398, 631, 1000 rpm

Speed diagram & kinematic layout: 2 × 3 arrangement

7. (a) Centrifugal clutch problem

Given:

P = 12 kW, N = 800 rpm, engagement at 600 rpm

r = 180 mm, CG = 140 mm, μ = 0.35, pressure = 0.3 N/mm²

Number of shoes = 4, θ = 70°
 Steps:

· Find centrifugal force at engagement speed

Use torque equation:

$$T = \mu \cdot R \cdot W$$

Determine shoe dimensions based on pressure and area

· Final answers: mass per shoe, shoe dimensions (length, width)

OR

(b) Single plate clutch power transmission

Given:

· Use torque formulas for:

Uniform pressure:

$$T = rac{2}{3} \mu F rac{r_o^3 - r_i^3}{r_o^2 - r_i^2}$$

Uniform wear:

$$T = \mu F \frac{r_o + r_i}{2}$$

• Power = $2\pi NT/60$

8. (a) 18-speed gearbox (1440 rpm → 16-800 rpm)

- Arrangement: 2×3×3
- Step ratio (R) ≈ 1.414
- Speeds: 16, 22.6, 32, ..., 800 rpm
- Sketch:
 - · Speed diagram: 3 layers
 - Kinematic layout: Cone + lay shaft + gears
- List of shaft speeds when output is 16 rpm: motor = 1440 rpm; intermediate speeds depend on gear ratios

OR

(b) Brake problem

Given:

- D = 1 m, T = 240 Nm, N = 400 rpm, μ = 0.32
- Angles = 35°, 100°, arms a = 800 mm, b = 150 mm, c = 25 mm
- Use block brake torque formulas:
 - · For CW and CCW rotation
- · Self-locking condition:

$$\mu \cdot r \geq rac{c}{a}$$

Calculate new c for self-locking