



# SNS COLLEGE OF TECHNOLOGY

(An Autonomous Institution)

Approved by AICTE, New Delhi, Affiliated to Anna University, Chennai

Accredited by NAAC-UGC with 'A++' Grade (Cycle III) &

Accredited by NBA (B.E - CSE, EEE, ECE, Mech & B.Tech.IT)

COIMBATORE-641 035, TAMIL NADU

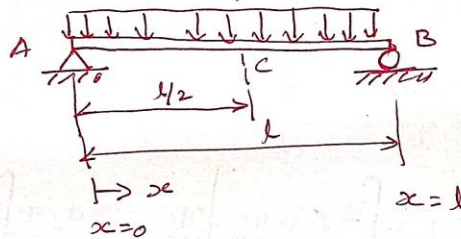


## DEPARTMENT OF AEROSPACE ENGINEERING

Faculty Name : **Dr.M.Subramanian,** Academic Year : **2024-2025 (Odd)**  
**Prof & Head/ Aerospace**  
 Year & Branch : **III Aerospace** Semester : **V**  
 Course : **19ASB302 – Finite Element Method for Aerospace**  
 Unit: 1

using trigonometric series - two term trial function

Derive the expression for deflection and bending moment in a simply supported beam of span length  $l$ , subjected to uniformly distributed load over entire span using two term trigonometric trial function. Also find the deflection and moment at mid span and compare with exact solution. Use Rayleigh-Ritz method. <sup>[AU 2008]</sup>  
 w/unit length



Two term trigonometric trial function such as

$$y = a_1 \sin \frac{\pi x}{l} + a_2 \sin \frac{3\pi x}{l} \quad \text{--- (1)}$$

The boundary condition for the uniformly distributed loaded beam are

$$y = 0 \quad \text{at } x=0 \quad \text{and} \quad \text{at } x=l$$

Since the above selected trial function satisfies the boundary conditions, this function is correct.

On applying the total potential energy concept we get,

$$\Pi = U - W$$

where  $U$  = Internal strain energy  
 $W$  = work done by the external force

①



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Strain energy for a beam is given by

$$U = \frac{EI}{2} \int_0^l \left( \frac{d^2y}{dx^2} \right)^2 dx \quad \text{--- (3)}$$

Now differentiating the deflection functions  
[ie. Equation (1)] two times we get

$$y = a_1 \sin \frac{\pi x}{l} + a_2 \sin \frac{3\pi x}{l}$$

$$\frac{dy}{dx} = a_1 \left[ \cos \frac{\pi x}{l} \right] \frac{\pi}{l} + a_2 \left[ \cos \frac{3\pi x}{l} \right] \frac{3\pi}{l}$$

$$= \frac{a_1 \pi}{l} \cos \frac{\pi x}{l} + \frac{3a_2 \pi}{l} \cos \frac{3\pi x}{l}$$

$$\frac{d^2y}{dx^2} = \frac{a_1 \pi}{l} \left[ -\sin \frac{\pi x}{l} \right] \frac{\pi}{l} + \frac{3a_2 \pi}{l} \left[ -\sin \frac{3\pi x}{l} \right] \frac{3\pi}{l}$$

$$= -\frac{a_1 \pi^2}{l^2} \sin \frac{\pi x}{l} - \frac{9a_2 \pi^2}{l^2} \sin \frac{3\pi x}{l}$$

Hence

$$\left[ \frac{d^2y}{dx^2} \right]^2 = \left[ -\frac{a_1 \pi^2}{l^2} \sin \frac{\pi x}{l} - \frac{9a_2 \pi^2}{l^2} \sin \frac{3\pi x}{l} \right]^2$$

$$= \left[ \frac{a_1 \pi^2}{l^2} \sin \frac{\pi x}{l} + \frac{9a_2 \pi^2}{l^2} \sin \frac{3\pi x}{l} \right]^2$$

$$= \frac{\pi^4}{l^4} \left[ a_1 \sin \frac{\pi x}{l} + 9a_2 \sin \frac{3\pi x}{l} \right]^2$$

(2)

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$$= \frac{\pi^4}{14} \left[ a_1^2 \sin^2 \frac{\pi x}{l} + 81 a_2^2 \sin^2 \frac{3\pi x}{l} + 18 a_1 a_2 \sin \frac{\pi x}{l} \sin \frac{3\pi x}{l} \right]$$

using the relation  $(a+b)^2 = a^2 + b^2 + 2ab$

Now, the strain energy equation (i.e. Eqn (3)) implies.

$$U = \frac{EI}{2} \int_0^l \left[ \frac{d^2 y}{dx^2} \right]^2 dx$$

$$= \frac{EI}{2} \int_0^l \frac{\pi^4}{14} \left[ a_1^2 \sin^2 \frac{\pi x}{l} + 81 a_2^2 \sin^2 \frac{3\pi x}{l} + 18 a_1 a_2 \sin \frac{\pi x}{l} \sin \frac{3\pi x}{l} \right] dx$$

$$= \frac{EI \pi^4}{2 \cdot 14} \int_0^l \left[ a_1^2 \sin^2 \frac{\pi x}{l} + 81 a_2^2 \sin^2 \frac{3\pi x}{l} + 18 a_1 a_2 \sin \frac{\pi x}{l} \sin \frac{3\pi x}{l} \right] dx$$

Since the simultaneous integration of all the three inner terms of equation (4) is slightly difficult, let us integrate

them individually

Now

$$\int_0^l a_1^2 \sin^2 \frac{\pi x}{l} dx = a_1^2 \int_0^l \frac{1}{2} \left[ 1 - \cos \frac{2\pi x}{l} \right] dx$$

using the relation  $\sin^2 x = \frac{1}{2}(1 - \cos 2x)$

$$= \frac{a_1^2}{2} \left[ x - \frac{\sin \frac{2\pi x}{l}}{\frac{2\pi}{l}} \right]_0^l$$

$$= \frac{a_1^2}{2} \left[ (l-0) - \frac{l}{2\pi} (\sin 2\pi - \sin 0) \right]$$

$$= \frac{a_1^2 l}{2}$$

(5)

$\sin 2\pi = 0$   
 $\sin 0 = 0$



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Similarly

$$\int_0^l 81a_2^2 \sin^2 \frac{3\pi x}{l} dx$$

$$= 81a_2^2 \int_0^l \frac{1}{2} \left[ 1 - \cos \frac{6\pi x}{l} \right] dx$$

$$= \frac{81a_2^2}{2} \left[ x - \frac{\sin \frac{6\pi x}{l}}{\frac{6\pi}{l}} \right]_0^l$$

$$= \frac{81a_2^2}{2} \left[ (l-0) - \frac{l}{6\pi} (\sin 6\pi - \sin 0) \right] \begin{array}{l} \sin 6\pi = 0 \\ \sin 0 = 0 \end{array}$$

$$= \frac{81a_2^2}{2} l \quad \text{--- (3)}$$

and

$$\int_0^l 18a_1 a_2 \sin \frac{\pi x}{l} \sin \frac{3\pi x}{l} dx$$

$$= 18a_1 a_2 \int_0^l \frac{\sin \frac{3\pi x}{l}}{l} \cdot \frac{\sin \frac{\pi x}{l}}{l} dx$$

$$= 18a_1 a_2 \int_0^l \frac{1}{8} \left( \cos \frac{2\pi x}{l} - \cos \frac{4\pi x}{l} \right) dx$$

using the relation  $\sin A \sin B$   
 $= \frac{1}{2} [\cos(A-B) - \cos(A+B)]$

(4)



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$$= 9a_1 a_2 \left[ \frac{\sin 2\pi x}{\left(\frac{2\pi}{l}\right)} - \frac{\sin 4\pi x}{\left(\frac{4\pi}{l}\right)} \right]_0^l$$

$$= 9a_1 a_2 \left[ \frac{l}{2\pi} (\overset{0}{\sin 2\pi} - \overset{0}{\sin 0}) - \frac{l}{4\pi} (\overset{0}{\sin 4\pi} - \overset{0}{\sin 0}) \right] = 0 \Rightarrow \text{⑥}$$

∴  
 $\sin 2\pi = 0$   
 $\sin 4\pi = 0$   
 $\sin 0 = 0$

Substituting the values of equation (5), (6) and (7) in equation (4), we get

$$U = \frac{EI\pi^4}{4l^3} \left[ \frac{a_1^2 l}{2} + \frac{81a_2^2 l}{2} \right]$$

$$= \frac{EI\pi^4}{4l^3} [a_1^2 + 81a_2^2]$$

Now, the work done by the external force is given by

$$W = \int_0^l w y dx$$

$$= \int_0^l w \left[ a_1 \sin \frac{\pi x}{l} + a_2 \sin \frac{3\pi x}{l} \right] dx$$

$$= w \left[ \int_0^l a_1 \sin \frac{\pi x}{l} dx + \int_0^l a_2 \sin \frac{3\pi x}{l} dx \right]$$

$$= w \left\{ \left[ a_1 \frac{-\cos \frac{\pi x}{l}}{\pi/l} \right]_0^l + \left[ a_2 \frac{-\cos \frac{3\pi x}{l}}{3\pi/l} \right]_0^l \right\}$$

⑤



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$$\begin{aligned} &= W \left[ -\frac{a_1 l}{\pi} (\cos \pi - \cos 0) - \frac{a_2 l}{3\pi} (\cos 3\pi - \cos 0) \right] \\ &= W \left[ -\frac{a_1 l}{\pi} (-1 - 1) - \frac{a_2 l}{3\pi} (-1 - 1) \right] \quad \left\{ \begin{array}{l} \therefore \cos \pi = -1 \\ \cos 3\pi = -1 \\ \cos 0 = 1 \end{array} \right\} \\ &= W \left[ \frac{2a_1 l}{\pi} + \frac{2a_2 l}{3\pi} \right] \\ &= \frac{2Wl}{\pi} \left( a_1 + \frac{a_2}{3} \right) \quad \text{--- (4)} \end{aligned}$$

Substituting the values of equation (2) & (4) in equation (3) we get,

Total potential energy,  $\pi = U - W$

$$= \frac{EI \pi^4}{4l^3} (2a_1 + 8a_2) - \frac{2Wl}{\pi} \left( a_1 + \frac{a_2}{3} \right)$$

For minimum potential energy, the following condition must be satisfied.

$$\frac{\partial \pi}{\partial a_1} = 0 \quad \text{and} \quad \frac{\partial \pi}{\partial a_2} = 0$$

$$\Rightarrow \frac{\partial \pi}{\partial a_1} = \frac{EI \pi^4}{4l^3} (2a_1) - \frac{2Wl}{\pi} = 0 \Rightarrow a_1 = \frac{4Wl^4}{EI \pi^5}$$

$$\frac{\partial \pi}{\partial a_2} = \frac{EI \pi^4}{4l^3} (8a_2) - \frac{2Wl}{3\pi} = 0 \Rightarrow a_2 = \frac{4Wl^4}{243EI \pi^5}$$

(6)



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once the deflection

$$y = a_1 \sin \frac{\pi x}{l} + a_2 \sin \frac{3\pi x}{l}$$

$$= \frac{4wl^4}{EI\pi^5} \sin \frac{\pi x}{l} + \frac{4wl^4}{243EI\pi^5} \sin \frac{3\pi x}{l}$$

Maximum deflection at  $x = l/2$  is obtained as

$$y_{\max} = \frac{4wl^4}{EI\pi^5} \sin\left(\frac{\pi}{l} \times \frac{l}{2}\right) + \frac{4wl^4}{243EI\pi^5} \sin\left(\frac{3\pi}{l} \times \frac{l}{2}\right)$$

$$= \frac{4wl^4}{EI\pi^5} \sin \frac{\pi}{2} + \frac{4wl^4}{243EI\pi^5} \sin\left(\frac{3\pi}{2}\right)$$

$$= \frac{4wl^4}{EI\pi^5} - \frac{4wl^4}{243EI\pi^5} \left[ \because \sin \frac{\pi}{2} = 1, \sin \frac{3\pi}{2} = -1 \right]$$

$$= \frac{4wl^4}{EI\pi^5} \left[ 1 - \frac{1}{243} \right]$$

$$= \frac{3.984 wl^4}{EI\pi^5}$$

$$= \frac{wl^4}{EI \left[ \frac{\pi^5}{3.984} \right]} = \frac{wl^4}{76.81 EI}$$

$$y_{\max} = \frac{wl^4}{76.81 EI}$$

This is almost equal to the exact value of maximum deflection which is

$$y_{\max} = \frac{5}{384} \frac{wl^4}{EI} = \frac{wl^4}{76.8 EI}$$

(7)



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Determination of bending moment at mid-span

For a beam, the bending moment is given by the expression as

$$\text{Bending moment, } M = EI \frac{d^2y}{dx^2}$$

From the previous derivation we know that

$$\frac{d^2y}{dx^2} = -\frac{a_1\pi^2}{l^2} \sin \frac{\pi x}{l} - \frac{9a_2\pi^2}{l^2} \sin \frac{3\pi x}{l}$$

Hence

$$M = EI \left( -\frac{a_1\pi^2}{l^2} \sin \frac{\pi x}{l} - \frac{9a_2\pi^2}{l^2} \sin \frac{3\pi x}{l} \right)$$

$$= -\frac{EI\pi^2}{l^2} \left( a_1 \sin \frac{\pi x}{l} + 9a_2 \sin \frac{3\pi x}{l} \right)$$

Substituting the values of  $a_1$  and  $a_2$  in the above equation, we get

$$M = -\frac{EI\pi^2}{l^2} \left( \frac{4wl^4}{E\pi^5} \sin \frac{\pi x}{l} + 9 \times \frac{4wl^4}{243E\pi^5} \sin \frac{3\pi x}{l} \right)$$
$$= -\left( \frac{EI\pi^2}{l^2} \right) \left( \frac{4wl^4}{E\pi^5} \right) \left[ \sin \frac{\pi x}{l} + \frac{9}{243} \sin \frac{3\pi x}{l} \right]$$

Bending moment at  $x = \frac{l}{2}$  is obtained as

Maximum bending moment

$$M_{\max} = -\frac{4wl^2}{\pi^3} \left[ \sin \left( \frac{\pi}{l} \times \frac{l}{2} \right) + \frac{9}{243} \sin \left( \frac{3\pi}{l} \times \frac{l}{2} \right) \right]$$
$$= -\frac{4wl^2}{\pi^3} \left[ \sin \frac{\pi}{2} + \frac{9}{243} \sin \frac{3\pi}{2} \right]$$

(8)

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$$= -\frac{4wl^2}{\pi^3} \left(1 - \frac{9}{243}\right) \quad \left[ \because \sin \frac{\pi}{2} = 1 \ \& \ \sin \frac{3\pi}{2} = -1 \right]$$

$$= -\frac{3.852wl^2}{\pi^3} = -\frac{wl^2}{\left(\frac{\pi^3}{3.852}\right)}$$

$$= -\frac{wl^2}{8.05}$$

Negative sign is due to downward loading

$$M_{\max} = \frac{wl^2}{8.05}$$

This is almost equal to the exact value of maximum bending moment which

$$M_{\max} = \frac{wl^2}{8}$$