



# SNS COLLEGE OF TECHNOLOGY

(An Autonomous Institution)

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ACADEMIC YEAR 2024-25

Department of Civil Engineering ( SA I)



## Unit - 1

Fundamental concepts and Energy Methods.

Determinate Structure :

The structure which can be solved with the available equilibrium equations ( $\sum V=0$ ,  $\sum H=0$ ,  $\sum M=0$ ) is known as a determinate structure.

Ex: Simply supported beam & cantilever beam.

Indeterminate Structure :

The structure which cannot be solved with the available equilibrium equations is known as indeterminate structures.

Ex: Continuous beam, Propped cantilever beam & fixed beams.

Consistent Deformation method for pin-jointed frames — Procedure.

Step 1: Calculate the indeterminacy.

$$I = I_i + I_e$$

Step-2: Calculate (F) forces of the members using method of joints for the actual external loads.

Step-3: Calculate k-forces of the members using method of joints by removing all external loads and apply unit force along the redundant.

Step-4: Calculate the redundant force using the formula.

$$T = \frac{-\sum \frac{PKL}{AE}}{\sum \frac{k^2L}{AE}}$$



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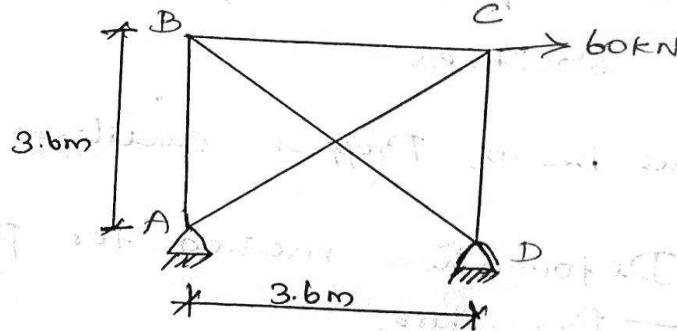


Step - 5: Calculate forces in other members, using formula.

$$P = F + kT$$

Step - 6: Do all the calculations in a tabular form.

1. Determine the forces in the members of the truss shown below by consistent deformation method.  $AE$  is constant for all the members.



Solution:

Step-1: Calculation of Indeterminacy:

$$I_i = m - (2j - 3) \quad [m = 5, j = 4]$$

$$I_i = 5 - (2 \times 4) + 3$$

$$I_i = 0$$

$$I_e = r - 3 = 4 - 3 = 1$$

Hence the structure is externally indeterminate to first degree. Replace the hinged support at D by a roller support so that the horizontal force is treated as the redundant.



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Step - 2 : Calculation of F-forces using method of joints:

At Joint A :

$$\sum M_A = 0 \Rightarrow$$

$$-V_D \times 3.6 + 60 \times 3.6 = 0$$

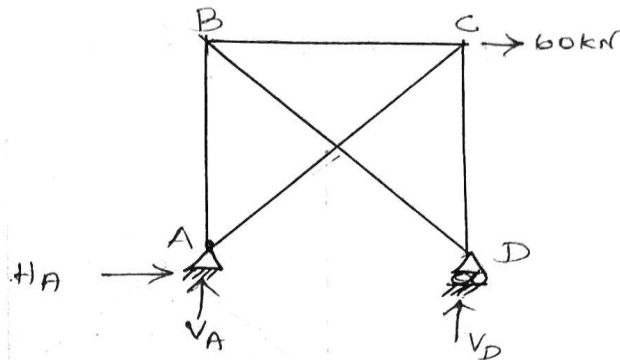
$$V_D = 60 \text{ kN } (\uparrow)$$

$$V_A = 60 \text{ kN } (\downarrow)$$

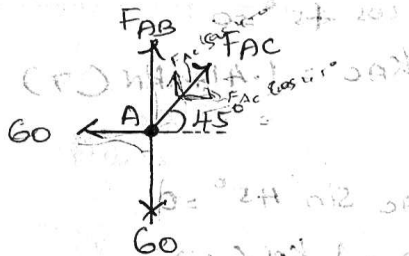
$$\sum H = 0,$$

$$+H_A - 60 = 0$$

$$H_A = 60 \text{ kN } (\leftarrow)$$



Joint A :



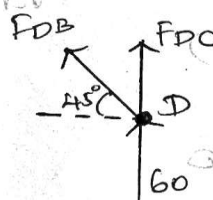
$$\sum H = 0 \Rightarrow -60 + F_{AC} \cos 45^\circ = 0$$

$$F_{AC} = 84.85 \text{ kN } (T)$$

$$\sum V = 0 \Rightarrow F_{AB} - 60 + F_{AC} \sin 45^\circ = 0$$

$$F_{AB} = 0$$

Joint D :



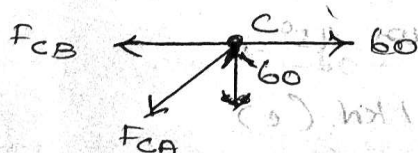
$$\sum H = 0 \Rightarrow -F_{DB} \cos 45^\circ = 0$$

$$F_{DB} = 0$$

$$\sum V = 0 \Rightarrow F_{DC} + 60 + F_{DB} \sin 45^\circ = 0$$

$$F_{DC} = 60 \text{ kN } (C)$$

Joint C :



$$\sum H = 0 \Rightarrow -F_{CB} - F_{CA} \cos 45^\circ + 60 = 0$$

$$F_{CB} = 0$$



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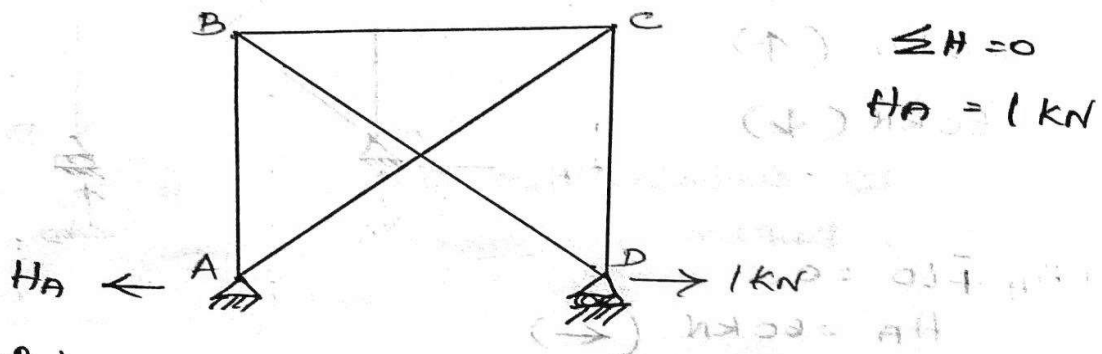
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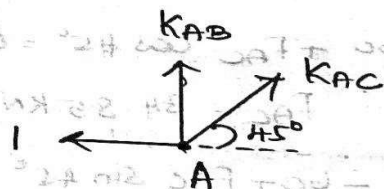


Step - 3 : Calculation of k - forces :

Remove all the external loads and apply unit force at D and use method of joints.



Joint A :



$$\sum H = 0$$

$$-1 + K_{AC} \cos 45^\circ = 0$$

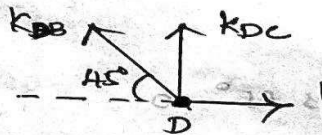
$$K_{AC} = 1.414 \text{ kN (T)}$$

$$\sum V = 0$$

$$K_{AB} + K_{AC} \sin 45^\circ = 0$$

$$K_{AB} = 1 \text{ kN (C)}$$

Joint D :



$$\sum H = 0$$

$$1 - K_{DB} \cos 45^\circ = 0$$

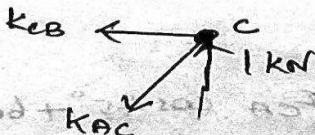
$$K_{DB} = 1.414 \text{ kN (T)}$$

$$\sum V = 0$$

$$K_{DC} - K_{DB} \sin 45^\circ = 0$$

$$K_{DC} = 1 \text{ kN (C)}$$

Joint C :



$$\sum H = 0,$$

$$-K_{CB} - 1.414 \cos 45^\circ = 0$$

$$K_{CB} = 1 \text{ kN (C)}$$

Horizontal  
- Cos  
Vertical  
- Sin





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Step - 4 : Tabulation :

S.No	Member	F (KN)	K (KN)	L	FKL	K <sup>2</sup> L	P = F + K.T
1.	AB	0	-1	3.6	0	3.6	26.53
2.	BC	0	-1	3.6	0	3.6	26.53
3.	CD	-60	-1	3.6	216	3.6	-33.47
4.	AC	84.85	1.414	5.09	610.69	10.18	47.34
5.	BD	0	1.414	5.09	0	10.18	-37.51
					826.69	31.16	

Hence horizontal reaction @ D,

$$T = \frac{-\sum FKL/AE}{\sum K^2L/AE}$$

$$= \frac{-826.69}{31.16}$$

$$= -26.53 \text{ kN}$$

$$H_D = 26.53 \text{ kN} (\leftarrow)$$

$$P_{AB} = 0 + (-1) \times (-26.53) = 26.53 \text{ kN}$$

$$P_{BC} = 0 + (-1) \times (-26.53) = 26.53 \text{ kN}$$

$$P_{CD} = -60 + (-1) \times (-26.53) = -33.47 \text{ kN}$$

$$P_{AC} = 84.85 + (1.414 \times -26.53) = 47.34 \text{ kN}$$

$$P_{BD} = 0 + (1.414 \times -26.53) = -37.51 \text{ kN}$$



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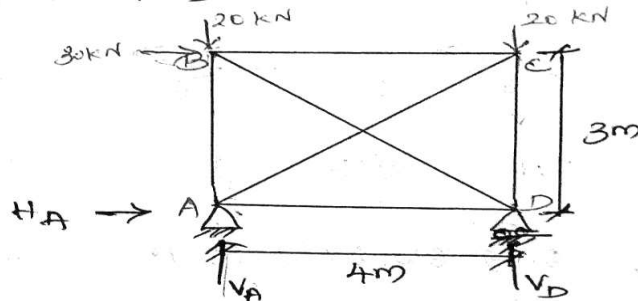
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2. Find the forces in all the members of the frame shown below in which the c/s Area of vertical members are  $300 \text{ mm}^2$  each, and those of all other members are  $2200 \text{ mm}^2$ .  $E = 2 \times 10^5 \text{ N/mm}^2$



Soln :

Step-1: Calculation of Indeterminacy :

$$I_i = m - 2j + 3$$
$$= 6 - (2 \times 4) + 3$$
$$= 1$$

$$I_e = r - 3 = 3 - 3 = 0$$

$$I = I_i + I_e = 1$$

Here the frame is internally indeterminate to degree 1. Hence member AC is removed to make the structure determinate.

Step-2: Calculation of F-forces :

$$\sum M_A = 0, \quad -R_D \times 4 + (20 \times 4) + (30 \times 3) = 0$$
$$R_D = 42.5 \text{ kN}$$

$$\sum V = 0, \quad R_A + R_D = 40$$

$$R_A + 42.5 = 40$$

$$R_A = -2.5 \text{ kN}$$

$$R_A = 2.5 \text{ kN} (\uparrow)$$

$$\sum H = 0, \quad 30 + H_A = 0$$

$$H_A = -30 \text{ kN}$$

$$H_A = 30 \text{ kN} (\leftarrow)$$



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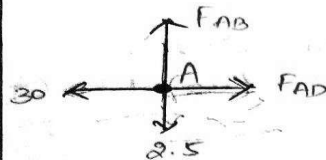
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Joint A :



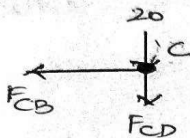
$$\sum H = 0, \quad -30 + F_{AD} = 0$$

$$F_{AD} = 30 \text{ kN (T)}$$

$$\sum V = 0, \quad F_{AB} - 2.5 = 0$$

$$F_{AB} = 2.5 \text{ kN (T)}$$

Joint C :



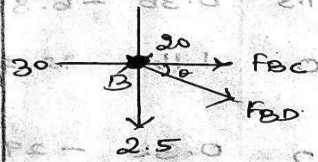
$$\sum H = 0, \quad F_{CB} = 0$$

$$\sum V = 0, \quad -20 - F_{CD} = 0$$

$$F_{CD} = -20 \text{ kN}$$

$$F_{CD} = 20 \text{ kN (C)}$$

Joint B :



$$\tan \theta = 3/4$$

$$\theta = \tan^{-1}(3/4) = 36^\circ 52'$$

$$\sum H = 0, \quad 30 + F_{BD} \cos 36^\circ 52' = 0$$

$$F_{BD} = -37.5 \text{ kN (T)}$$

$$F_{BD} = 37.5 \text{ kN (C)}$$

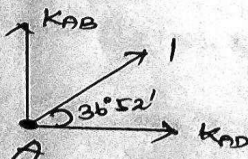
$$\sum V = 0, \quad -20 - 2.5 - F_{BD} \sin 36^\circ 52' = 0$$

$$F_{BD} = 37.5 \text{ kN (C)}$$

Step 31: Calculation of k-forces :

Remove all the external forces & apply unit force along AC.

Joint A :

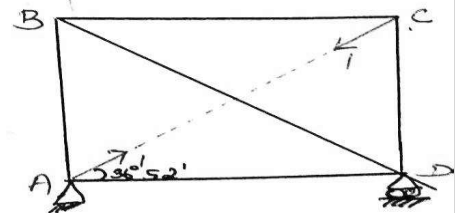


$$\sum H = 0, \quad K_{AD} + 1 \cos 36^\circ 52' = 0$$

$$K_{AD} = 0.8 \text{ kN (C)}$$

$$\sum V = 0, \quad K_{AB} + 1 \sin 36^\circ 52' = 0$$

$$K_{AB} = 0.6 \text{ kN (C)}$$





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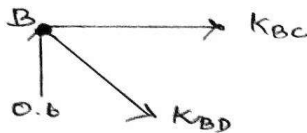
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Joint B :



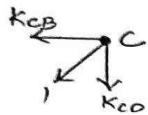
$$\sum H = 0, K_{BC} + K_{BD} \cos 36^\circ 52' = 0$$

$$K_{BC} = 0.8 \text{ kN (C)}$$

$$\sum V = 0, 0.6 - K_{BD} \sin 36^\circ 52' = 0$$

$$K_{BD} = 1 \text{ kN (T)}$$

Joint C :



$$\sum V = 0, -K_{CD} - 1 \sin 36^\circ 52' = 0$$

$$K_{CD} = 0.6 \text{ kN (C)}$$

Step - 4: Tabulation :

sl.no	Member	F (KN)	K (KN)	l (mm)	A (mm <sup>2</sup> )	$\frac{FKl}{A}$	$\frac{k^2l}{A}$	$P = F + kT$
1.	AB	2.5	-0.6	3000	3000	-1.5	0.36	-6.866
2.	BC	0	-0.8	4000	2200	0	-1.16	-12.488
3.	CD	-20	-0.6	3000	3000	12	0.36	-29.366
4.	DA	30	-0.8	4000	2200	-43.63	1.16	17.512
5.	BD	-37.5	-1.0	5000	2200	-85.23	2.27	-21.89
6.	AC	0	1.0	5000	2200	0	2.27	15.61
						-118.36	7.58	

Force in the redundant AC is

$$T = - \frac{\sum FKl/AE}{\sum k^2l/AE}$$

$$= - \frac{(-118.36)}{7.58}$$

$$T = 15.61 \text{ kN (T)}$$

$$P_{AB} = -6.866 \text{ kN}$$

$$P_{DA} = 17.512 \text{ kN}$$

$$P_{BC} = -12.488 \text{ kN}$$

$$P_{BD} = -21.89 \text{ kN}$$

$$P_{CD} = -29.366 \text{ kN}$$

$$P_{AC} = 15.61 \text{ kN}$$



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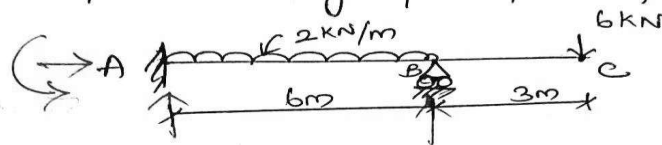
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Principle of Superposition :

The Principle of Super position simply states that on a linear elastic structure, the combined effect of several loads acting simultaneously is equal to the algebraic sum of the effects of each load acting individually.

3 Solve the problem by principle of superposition.



$$R = 4 \\ r = 3$$

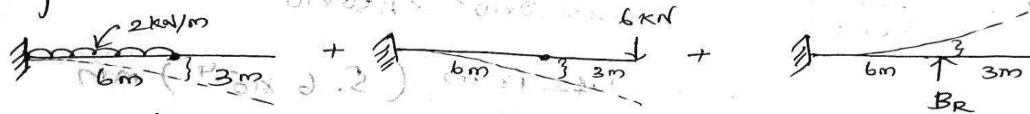
Soln :

Step-1: Find indeterminacy :

$$D_s = R - r \\ = 4 - 3 = 1$$

Reaction at B is considered as redundant.

Step-2: Deflection of the beam:



$$\frac{-wL^4}{8EI} - \frac{PL^3}{3EI} + \frac{PL^3}{3EI} = 0$$

$$-\frac{2 \times 6^4}{8EI} - \frac{6 \times 9^3}{3EI} + \frac{B_R \times 6^3}{3EI} = 0$$

$$-\frac{2 \times 6^4}{8} - \frac{6 \times 9^3}{3} + \frac{B_R \times 6^3}{3} = 0$$

$$-324 - 1458 + 72 B_R = 0$$

$$B_R = 24.75 \text{ kN}$$

$$\sum V = 0, \quad 24.75 - 6 - (6 \times 2) + V_A = 0$$

$$V_A = -6.75 \text{ kN}$$

$$\sum M_A = 0,$$

$$M_A - (6 \times 2 \times 6/2) + (24.75 \times 6) - (6 \times 9) = 0$$

$$M_A + 58.5 = 0$$

$$M_A = -58.5 \text{ kNm}$$

$$M_A + (24.75 \times 6) - (6 \times 9) + 12 \times \frac{6}{2} = 0$$



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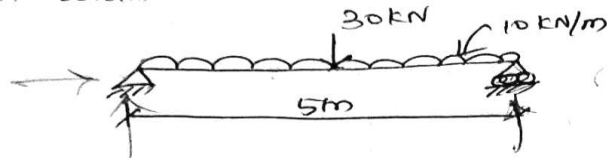
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4. Find deflection at centre of simply supported beam as shown below, it is a rectangular section with width 200mm and depth 350mm.  $E = 2 \times 10^5$



$$R = 3$$

$$r = 3$$

Soln :

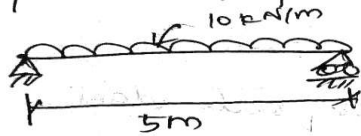
Step - 1 : Find indeterminacy :

$$D_s = R - r$$

$$= 3 - 3 = 0$$

Statically determinate structure.

Step - 2 : Deflection of the beam :

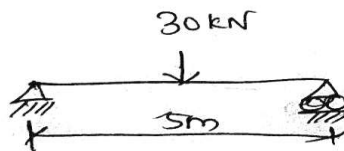


$$I = \frac{bd^3}{12} = \frac{200 \times 350^3}{12}$$

$$= 450 \times 10^6$$

$$\delta_1 = \frac{5WL^4}{384EI} = \frac{5 \times 10 \times 5000^4}{384 \times 2 \times 10^5 \times 450 \times 10^6}$$

$$= 0.9042 \text{ mm } (5.6 \times 10^{-4}) \text{ mm}$$



$$\delta_2 = \frac{WL^3}{48EI} = \frac{30 \times 1000 \times 5000^3}{48 \times 2 \times 10^5 \times 450 \times 10^6} = 0.8681 \text{ mm}$$

$$\text{Total deflection} = 0.9042 + 1.7723 \text{ mm}$$

Degree of Freedom:

The Number of Independent joint Displacement in a structure is termed as Degree of Freedom. Kinematic Indeterminacy: Associated with DOF of structure.



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## Methods of Structural Analysis :

### 1. Force Method :

- \* Used to analyse the indeterminate structures.
- \* Used to compute internal forces and reactions.
- \* Frames of single storey and uncommon geometry such as gabled frame.

### 2. Displacement Method :

- \* Unknown displacements are taken as terms of load - displacement relationship.
- \* By solving equilibrium equations the unknown displacement and loads are determined.

### 2-1 Slope Deflection Method :

- \* Used to analyse statically determinate and indeterminate beams and frames.
- \* All deformations are assumed only due to bending only.
- \* All the joints of the frames are rigid, the angles between the members do not change.

### 2-2 Moment Distribution Method :

- \* It is an approximation method used for any desired degree of accuracy.
- \* In this method each joint of the structure is fixed.

### 2-3 Direct Stiffness method :

- \* This is a matrix analysis method, equilibrium equations are formulated to a single matrix relationship.





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## 3. Approximate Methods :

- \*. The forces and moments determined by this method are approximate values.
- \*. Based on realistic assumptions.

### 3.1 Portal method :

- \*. Used to analyse frames subjected to horizontal loads.
- \*. Points of inflections are assumed to be located at mid height of the column.
- \*. For beams it is at mid span.

### 3.2 Cantilever method :

- \*. Used for high rise structures.
- \*. Inflection point occurs at mid point of each girder.

### 3.3 Points of Inflection method :

- \*. Used to analyse the frames subjected to vertical loads.
- \*. Frame is reduced to statically determinate form by introducing no. of points of inflection.
- \*. The loading on the frame is usually UDL.
- \*. Axial forces are negligible.

## 4. Kani's Method :

- \*. In this method unknown fixed end moments are distributed to adjacent joints.
- \*. It is a self corrective method. The errors at any stage of iteration is corrected in subsequent steps.



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Energy Methods :

1. Virtual Work principle
2. Castigliano's Principle

Principle of Virtual work :

The work done by all the forces acting on a system, during a small virtual displacement is zero.

Castigliano's first theorem :

The first partial derivative of the total internal energy (strain energy) in a structure with respect to any particular deflection component at a point is equal to the force applied at that point and in the direction corresponding to that deflection component.

$$P = \frac{\partial U}{\partial \delta}$$

Applicable for linearly or non-linearly elastic structures for which the temperature is constant and the supports are unyielding.

Castigliano's second theorem :

The first partial derivative of the total internal energy in a structure with respect to the force applied at any point is equal to the deflection at the point of application of that force in the direction of its line of action.

$$\delta = \frac{\partial U}{\partial P}$$

Applicable to linearly elastic structures with constant temperature and unyielding supports.



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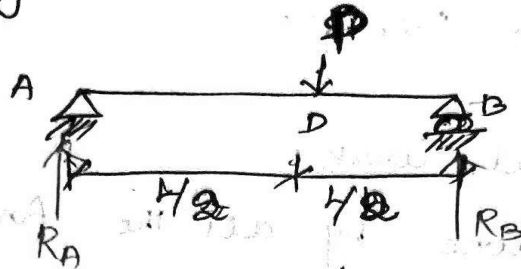
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5. Calculate the deflection of a given beam by Castigliano's Theorem.



Solution:

Step-1: Unknown Reactions:

$$\sum V = 0, R_A + R_B = P$$

$$\sum M_A = 0, -R_B \times 1 + P \times \frac{1}{2} = 0$$

$$R_B = \frac{P}{2}$$

$$R_A = \frac{P}{2}$$

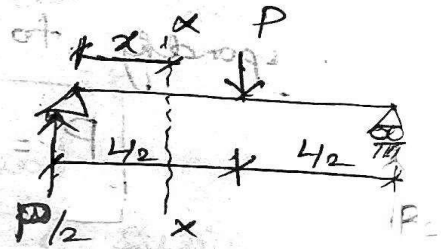
Step-2: Strain Energy:

Consider a section x-x at a distance x from A,

$$M_x = \frac{P}{2} x$$

$$U = \int \frac{M^2 dx}{2EI}$$

$$= 2 \int_0^{1/2} \left( \frac{P}{2} x \right)^2 dx$$





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By second theorem,

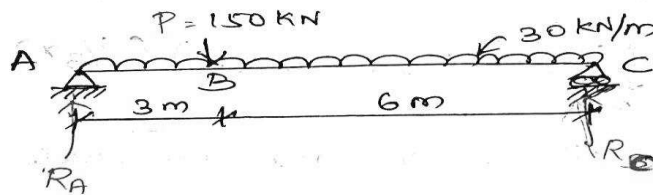
$$\delta = \frac{\partial U}{\partial w}$$

$$\delta_c = \frac{\partial (P^2 l^3)}{96EI} \frac{\partial}{\partial P}$$

$$= \frac{2Pl^3}{96EI}$$

$$\delta_c = \frac{Pl^3}{48EI} \quad (\text{or}) \quad \frac{Wl^3}{48EI}$$

6. Determine the vertical deflection at point load 150 kN as shown in figure. Take  $E = 232 \times 10^6 \text{ kN/m}^2$  and  $I = 8.789 \times 10^4 \text{ m}^4$ .



Soln :

Step-1 : Unknown Reactions :

$$\begin{aligned} \sum v = 0, \quad R_A + R_B &= 150 + (30 \times 9) \\ &= 150 + 270 \quad (\text{or}) \quad \boxed{P + 270} \\ &= 420 \text{ kN} \end{aligned}$$

$$\begin{aligned} \sum M_A = 0, \quad 9R_C - (30 \times 9 \times \frac{9}{2}) - (150 \times 3) &= 0 \\ 9R_C - (30 \times 9 \times \frac{9}{2}) - R_B &= 0 \\ 9R_C &= 1665 \text{ kN} \quad (\text{or}) \quad R_C = \left(\frac{P}{3} + 135\right) \text{ kN} \end{aligned}$$

$$\begin{aligned} -9R_C + 30 \times 9 \times \frac{9}{2} + 150 \times 3 &= 0 \\ -9R_C + 1215 + 450 &= 0 \\ -9R_C &= -1665 \\ R_C &= 185 \text{ kN} \end{aligned}$$

$$R_C = 185 \text{ kN}$$

$$R_A = 235 \text{ kN}$$

$$R_A = \left(\frac{2}{3}P + 135\right) \text{ kN}$$



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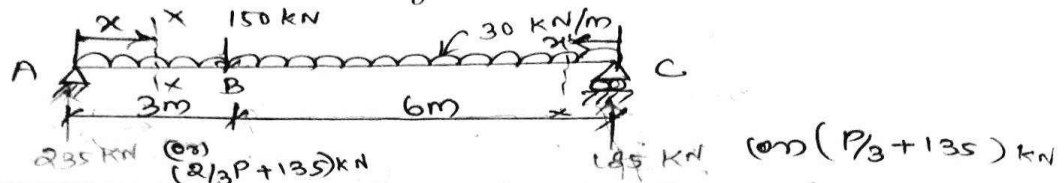
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Step - 2: Strain Energy :



Segment	Origin	Limits	Moment	$\frac{\partial M}{\partial P}$
AB	A	0 to 3m	$235x - (30 \times x \times \frac{x}{2})$ $\frac{2}{3}Px + 135x - \frac{30x^2}{2}$	$\frac{2}{3}x$
CB	C	0 to 6m	$\frac{P}{3}x + 135x - \frac{30x^2}{2}$	$\frac{1}{3}x$

$$\delta_B = \frac{1}{EI} \int_0^L M \left( \frac{\partial M}{\partial P} \right) dx$$

$$= \frac{1}{EI} \left( \int_0^3 \left( \frac{2P}{3}x + 135x - 15x^2 \right) \frac{2}{3}x \cdot dx + \int_0^6 \left( \frac{P}{3}x + 135x - 15x^2 \right) \left( \frac{1}{3}x \right) dx \right)$$

substitute  $P = 150 \text{ kN}$  and integrate,

$$= \frac{1}{EI} \left( \int_0^3 \left( \frac{300}{3}x + 135x - 15x^2 \right) \frac{2}{3}x \cdot dx + \int_0^6 \left( \frac{150}{3}x + 135x - 15x^2 \right) \frac{1}{3}x \cdot dx \right)$$

$$= \frac{1}{EI} \left( \int_0^3 (235x - 15x^2) \frac{2}{3}x \cdot dx + \int_0^6 (185x - 15x^2) \frac{1}{3}x \cdot dx \right)$$



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$$\begin{aligned} \delta_B &= \frac{1}{EI} \left( \int_0^3 (156.67x^2 - 10x^3) dx + \int_0^6 (61.67x^2 - 5x^3) dx \right) \\ &= \frac{1}{EI} \left[ \left( \frac{156.67x^3}{3} - \frac{10x^4}{4} \right) \Big|_0^3 + \left( \frac{61.67x^3}{3} - \frac{5x^4}{4} \right) \Big|_0^6 \right] \\ &= \frac{1}{EI} \left[ (1410.03 - 202.5) + (4440.24 - 1620) \right] \\ &= \frac{4027.77}{EI} \end{aligned}$$

$$\delta_B = \frac{4027.77}{232 \times 10^6 \times 8.989 \times 10^{-4}}$$

$$= 0.01975 \text{ m}$$

$$= 19.75 \text{ mm}$$

(or)

Strain Energy  $U = \int \frac{M^2 dx}{2EI}$

$$\begin{aligned} U &= \frac{1}{2EI} \int_0^3 \left( \frac{2}{3} Px + 135x - \frac{30x^2}{2} \right)^2 dx + \\ &\quad \frac{1}{2EI} \int_0^6 \left( \frac{P}{3} x + 135x - \frac{30x^2}{2} \right)^2 dx \end{aligned}$$

$$\begin{aligned} \delta &= \frac{\partial U}{\partial W} = \frac{\partial}{\partial W} \int \frac{M^2 dx}{2EI} = \int \frac{\partial M}{\partial W} \cdot \frac{\partial M}{\partial W} \cdot \frac{dx}{2EI} \\ &= \frac{\partial}{\partial W} \int M \cdot \frac{\partial M}{\partial W} dx \end{aligned}$$



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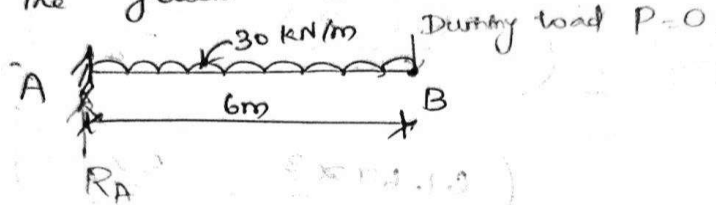
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7 By using Castigliano's theorem find deflection B for the given cantilever beam.



Soln:

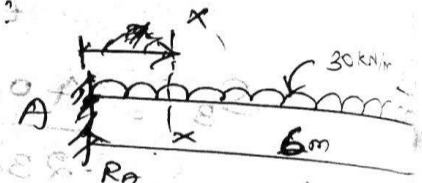
Step-1: Unknown Reactions:

$$R_A = (30 \times 6) + P$$

$$R_A = (180 + P) \text{ kN}$$

Step-2: Deflection under B

$$\delta_B = \frac{1}{EI} \int_0^L M \frac{\partial M}{\partial P} \cdot dx$$



Consider a section x-x at x distance from A

$$M_x = R_A x - (30 \times x \times x/2)$$

$$= (180 + P) x - (30 \times x^2/2)$$

Segment	Origin	Limits	Moment	$\frac{\partial M}{\partial P}$
AB	A	0 to 6m	$(180 + P)x - \frac{30x^2}{2}$ $= 180x - 15x^2 + Px$	$-x$

$$\delta_B = \frac{1}{EI} \int_0^6 (180x - 15x^2 + Px) \cdot (-x) \cdot dx$$

$$= \frac{1}{EI} \int_0^6 (180x^2 - 15x^3 + Px^2) \cdot dx$$





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$$\delta_B = \frac{1}{EI} \left[ \frac{180 \times 3}{3} + \frac{15 \times 4}{4} + \frac{Px^3}{3} \right]_0^6$$

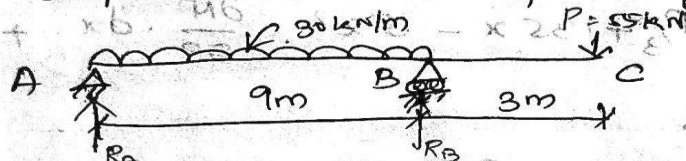
$$= \frac{1}{EI} [12 \times 180 + 4 \times 60 + 72P]$$

$$= \frac{4860}{EI} + 72P$$

Put  $P=0$  (dummy load)

$$\delta_B = \frac{4860}{EI}$$

8. Find deflection at C of the given beam.



$EI = \text{constant}$ ,  $E = 200 \text{ GPa}$ ,  $I = 830 \times 10^6 \text{ mm}^4$

Soln:

Step - 1: Determine x forces

$$R_A + R_B = P + 270$$

$$\sum M_A = 0, R_B \times 9 - (30 \times 9 \times \frac{9}{2}) - (P \times 12) = 0$$

$$R_B = \frac{4P}{3} + 135$$

$$\sum V = 0, R_A + R_B = P + 270$$

$$R_A = (P + 270) - \left( \frac{4P}{3} + 135 \right)$$

$$= \frac{P}{3} + 135$$



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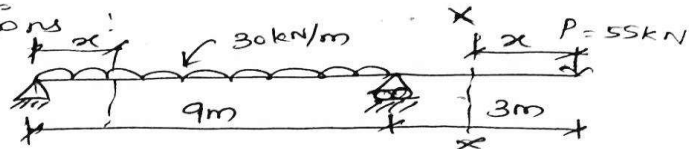
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Step - 2 : Deflections



Segment	Origin	Limits	Moment	$\frac{\partial M}{\partial P}$
AB	A	0 to 9	$(\frac{P}{3} + 135)x - 15x^2$	$x/3$
BC	C	0 to 3	$-Px$	$-x$

$$\delta_B = \int_0^L M \cdot \frac{\partial M}{\partial P} \cdot \frac{dx}{2EI}$$

$$= \frac{1}{2EI} \int_0^9 (P/3 + 135)x - 15x^2 \cdot \frac{\partial M}{\partial P} \cdot dx +$$

$$\frac{1}{2EI} \int_0^3 -Px \cdot \frac{\partial M}{\partial P} \cdot dx$$

$$= \frac{1}{2EI} \left[ \int_0^9 (P/3 + 135)x - 15x^2 \cdot x/3 \cdot dx + \right.$$

$$\left. \int_0^3 -Px(-x) \cdot dx \right]$$

$$\delta_B = \text{K. 54 mm}$$

$$= \frac{1}{2EI} \left[ \int_0^9 \left( \frac{Px^2}{9} + \frac{135x^2}{3} - \frac{15x^3}{3} \right) dx + \int_0^3 Px^2 \cdot dx \right]$$

$$= \frac{1}{2EI} \left[ \left( \frac{Px^3}{27} + \frac{135x^3}{9} - \frac{15x^4}{9} \right)_0^9 + \left( \frac{Px^3}{3} \right)_0^3 \right]$$

put  $P=55$



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$$\delta_B = \frac{1}{2EI} \left[ \frac{150 \times 9^3}{27} + \frac{135 \times 9^3}{9} - \frac{15 \times 9^4}{9} + \frac{135 \times 9^3}{9} \right]$$

$$= \frac{1}{2EI} (4050 + 10935 - 10935 + 495)$$

$$E = 200 \times 10^9 \text{ N/m}^2$$

$$= 200 \times 10^6 \text{ kN/m}^2$$

$$I = 830 \times 10^6 \text{ mm}^4$$

$$= 830 \times 10^6 \times 10^{-12} \text{ m}^4$$

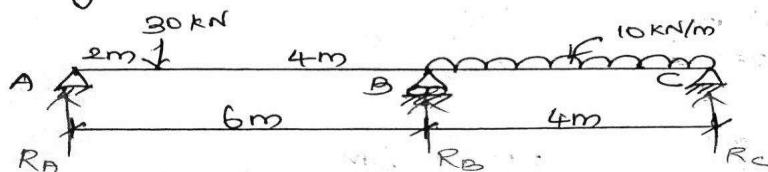
$$= 830 \times 10^{-6} \text{ m}^4$$

$$\delta_B = \frac{1}{200 \times 10^6 \times 830 \times 10^{-6}} (4545)$$

$$= 0.021 \text{ m}$$

$$\delta_B = 21 \text{ mm}$$

9. Analyse the continuous beam loaded as shown in figure by castigliano's method.



Soln:

Step - 1: Static Indeterminacy:

$$D_s = R - r$$

$$= 3 - 2$$

$$D_s = 1$$

Here vertical loadings only acting

$R_B$  is treated as a redundant. so remove

the support and treated as force.





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$$\sum \text{MORA} = 240 + 80 - 4R_B$$

$$R_A = 32 - 0.4R_B$$

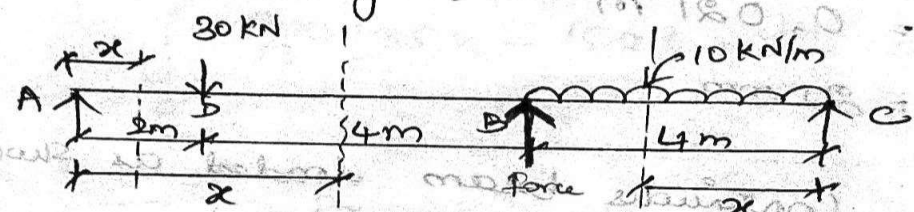
$$\sum V = 0 \quad R_C + R_A = 30 + (10 \times 4) - R_B$$

$$R_C + R_A = 70 - R_B$$

$$R_C = 70 - 32 + 0.4R_B - R_B$$

$$R_C = 38 - 0.6R_B$$

Step - 3: Calculating Redundant:



$$R_A = 32 - 0.4R_B$$

$$R_C = 38 - 0.6R_B$$

$$U = \int_0^l \frac{M^2 dx}{2EI} \quad \frac{\partial U}{\partial R_B} = 0$$

$$\frac{\partial}{\partial R_B} \int_0^l \frac{M^2 dx}{2EI} = \int_0^l \frac{2M}{2EI} \cdot \frac{\partial M}{\partial R_B} \cdot dx = 0$$

$$\frac{1}{EI} \int_0^l M \cdot \frac{\partial M}{\partial R_B} \cdot dx = 0$$

$$\int_0^l M \cdot \frac{\partial M}{\partial R_B} \cdot dx = 0$$



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Segment	Origin	Limits	Moment	$\frac{\partial M}{\partial R_B}$
AD	A	0 to 2	$(32 - 0.4R_B)x = 32x - 0.4R_B \cdot x$	$-0.4x$
DB	A	2 to 6	$(32 - 0.4R_B)x - 30(x-2)$ $= 32x - 0.4R_Bx - 30x + 60$ $= 2x - 0.4R_Bx + 60$	$-0.4x$
CB	C	0 to 4	$(38 - 0.6R_B)x - 10 \cdot x \cdot x / 2$ $= 38x - 0.6R_Bx - 5x^2$	$-0.6R_Bx$

$$\int_0^1 M \cdot \frac{\partial M}{\partial R_B} \cdot dx = 0$$

$$\int_0^2 (32x - 0.4R_B \cdot x) (-0.4x) dx + \int_2^6 (2x - 0.4R_Bx + 60) (-0.4x) dx +$$

$$\int_0^4 (38x - 0.6R_Bx - 5x^2) (-0.6x) dx = 0$$

$$\int_0^2 (-12.8x^2 + 0.16x^2 R_B) dx + \int_2^6 (-0.8x^2 + 0.16x^2 R_B - 24x) dx$$

$$+ \int_0^4 (-22.8x^2 + 0.36x^2 R_B + 3x^3) dx$$

$$\left( -12.8 \frac{x^3}{3} + 0.16 \frac{x^3}{3} R_B \right)_0^2 + \left( -0.8 \frac{x^3}{3} + 0.16 \frac{x^3}{3} R_B - \frac{24x^2}{2} \right)_2^6$$

$$+ \left( -22.8 \frac{x^3}{3} + 0.36 \frac{x^3}{3} R_B + 3 \frac{x^4}{4} \right)_0^4 = 0$$





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$$-34.133 + 0.427 R_B + [(-57.6 + 11.52 R_B - 432) - (-2.133 + 0.427 R_B - 48)] - (486.4 + 7.68 R_B + 192) - 768 + 19.2 R_B = 0$$

$$R_B = 40 \text{ kN}$$

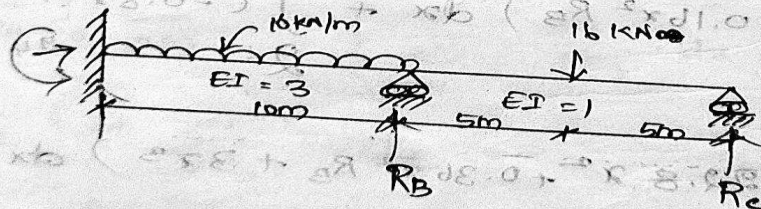
$$R_A = 32 - 0.4 R_B = 32 - (0.4 \times 40) = 16 \text{ kN}$$

$$R_C = 38 - 0.6 R_B = 38 - (0.6 \times 40) = 14 \text{ kN}$$

$$M_B = R_A \times 6 - 30 \times 4 = 16 \times 6 - 120 = 24 \text{ kNm}$$

$$M_B = R_C \times 4 - 10 \times 4 \times 4/2 = (14 \times 4) - (10 \times 8) = 24 \text{ kNm}$$

10. Analyse the continuous beam loaded as shown in fig. by Castigliano's theorem.



Soln :

Step-1 : Static Indeterminacy :

$$D_s = R - r$$

$$= 4 - 2$$

$$= 2$$

(Vertical loads only acting)

Consider  $R_B$  and  $R_C$  are redundant and remove the supports at B and C. As  $R_B$  and  $R_C$  are considered as forces.



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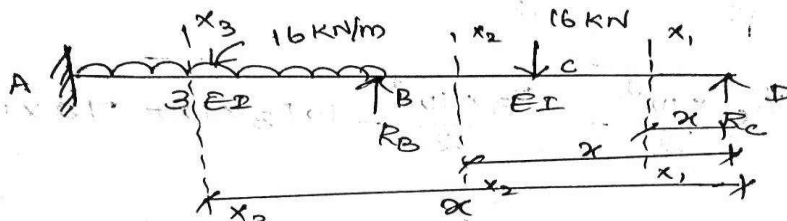
Step - 2 : Reactions :

$$U = \int M^2 \frac{dx}{2EI}$$

$$\frac{\partial U}{\partial R_B} = \int \frac{\partial M}{\partial EI} \frac{\partial M}{\partial R_B} \cdot dx = 0$$

$$\int_0^L \frac{M}{EI} \cdot \frac{\partial M}{\partial R_B} \cdot dx = 0 \rightarrow \textcircled{1}$$

$$\frac{\partial U}{\partial R_C} = 0 \Rightarrow \int \frac{M}{EI} \frac{\partial M}{\partial R_C} = 0 \rightarrow \textcircled{2}$$



Segment	Origin	Limits	Moment	$\frac{\partial M_x}{\partial R_B}$	$\frac{\partial M_x}{\partial R_C}$	EI
CD	C	0 to 5	$R_C \times x$	0	x	1
DB	C	5 to 10	$R_C x - 16(x-5)$	0	x	1
BA	C	10 to 20	$R_C x - 16(x-5) + R_B(x-10) - \frac{16(x-10)^2}{2}$	x-10	x	3

$$\textcircled{1} \Rightarrow \frac{\partial U}{\partial R_B} = 0 \Rightarrow \int_0^L \frac{M}{EI} \cdot \frac{\partial M}{\partial R_B} \cdot dx = 0$$

$$0 + 0 + \frac{1}{3} \int_{10}^{20} R_C x - 16(x-5) + R_B(x-10) - \frac{16(x-10)^2}{2} \cdot dx = 0$$

$$\frac{1}{3} \int_{10}^{20} R_C x - 16x + 80 + R_B x - 10R_B - \frac{16(x^2 + 100 - 20x)}{2} \cdot dx = 0$$





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$$\frac{1}{3} \int_0^{20} (R_C x - 16x + 80 + R_B x - 10R_B - Bx^2 - 800 + 160x) dx$$

$$\frac{1}{3} \int_0^{20} (R_C x + R_B x - 10R_B + 144x - 720) dx = 0$$

$$\frac{1}{3} \left[ \frac{R_C x^2}{2} + \frac{R_B x^2}{2} - 10R_B x + 144x^2 - 720x - R_C x \cdot 10 + 10R_B x + 100R_B + 144x^2 - 720x \right]_0^{20}$$

$$\frac{1}{3} \left[ \frac{R_C \times 20^2}{2} + \frac{R_B \times 20^2}{2} - 10R_B \times 20 + 72 \times 20^2 - 720 \times 20 \right]$$

$$\frac{R_C \times 10^2}{2} + \frac{R_B \times 10^2}{2} - 10R_B \times 10 + 72 \times 10^2 - 720 \times 10$$

$$\frac{1}{3} \left[ 200R_C + 200R_B - 200R_B + 28800 - 14400 - \right]$$

$$50R_C - 50R_B + 100R_B - 7200 + 7200$$

$$\frac{1}{3} \left[ 200R_C - 50R_C + 100R_B - 50R_B + 28800 - 14400 \right]$$

$$\frac{1}{3} \left[ 150R_C + 50R_B + 28800 \right] = 0$$

$$\frac{150R_C}{3} + \frac{50R_B}{3} + \frac{28800}{3} = 0$$

$$50R_C + 16.67R_B + 9600 = 0$$

$$-R_B + 2.5R_C = -87.99 \rightarrow \textcircled{3}$$



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$$\frac{\partial U}{\partial R_C} = 0 \Rightarrow \int_0^L \frac{M}{EI} \frac{\partial M}{\partial R_C} dx = 0$$

$$\frac{1}{EI} \int_0^5 R_C x \cdot x \cdot dx + \int_5^{10} R_C x - 16(x-5) \cdot x \cdot dx +$$

$$\frac{1}{3EI} \int_{10}^{20} R_C x - 16(x-5) + R_B(x-10) - 8(x-10)^2 \cdot x \cdot dx = 0$$

$$\frac{1}{EI} \int_0^5 R_C x^2 \cdot dx + \int_5^{10} R_C x \cdot x - 16x(x-5) \cdot dx +$$

$$\frac{1}{3EI} \int_{10}^{20} R_C x - 16x + 80 + R_B x - 10R_B - 8x(x-10)^2 \cdot dx = 0$$

$$\frac{1}{EI} \int_0^5 R_C x^2 \cdot dx + \int_5^{10} (R_C x^2 - 16x^2 + 80x) dx +$$

$$\frac{1}{3EI} \int_{10}^{20} R_C x - 16x + 80 + R_B x - 10R_B - 8x(x^2 + 100 - 20x) dx = 0$$

$$\frac{1}{EI} \int_0^5 R_C x^2 \cdot dx + \int_5^{10} (R_C x^2 - 16x^2 + 80x) dx +$$

$$\frac{1}{3EI} \int_{10}^{20} R_C x - 16x + 80 + R_B x - 10R_B - 8x^3 + 800x + 160x^2 dx = 0$$

$$\frac{1}{EI} \left[ R_C \frac{x^3}{3} \right]_0^5 + \left[ R_C \frac{x^3}{3} - \frac{16x^3}{3} + 80 \frac{x^2}{2} \right]_5^{10} +$$

$$\frac{1}{3EI} \left[ R_C \frac{x^2}{2} - 16x^2 + 80x + R_B \frac{x^2}{2} - 10R_B x - \frac{8x^4}{4} + 800x^2 + 160x \frac{x^3}{3} \right]_{10}^{20}$$



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$$\frac{1}{EI} \left( R_C \times 41.67 + R_C \times 333.33 - 5333.33 + 41.67R_C + 666.67 - 1000 \right) +$$

$$\frac{1}{3EI} \left( 200R_C - 6400 + 1600 + 200R_B - 200R_B - 32000 + 16000 + 42666.67 - 50R_C - 1600 + 800 + 50R_B - 2000 - 40000 + 5333 \right) = 0$$

$$R_B + 4R_C = 92.4 \quad \text{--- (A)}$$

$$R_B = 80.64 \text{ kN}$$

$$R_C = 2.94 \text{ kN}$$

BM @ CD,

$$M_x = R_C \times x$$

$$= 2.94 \times x$$

$$x=0, M_c = 0$$

$$x=5, M_D = 14.75 \text{ kNm}$$

$$R_A = (16 \times 10) + 16 - R_B - R_C$$

$$= 160 + 16 - 80.64 - 2.94$$

$$R_A = 92.42 \text{ kN}$$



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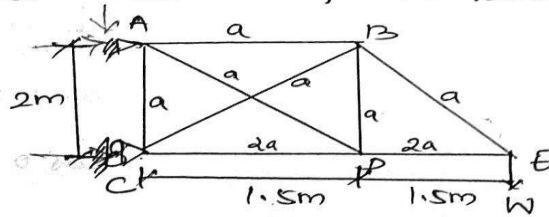
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11. Analyse the truss by Castigliano's theorem. The members are all of the same material.



Soln :

Step-1: Static indeterminacy :

$$m = 8, j = 5, R = 3$$

$$D_s = m + R - 2j$$

$$= (8 + 3) - (2 \times 5)$$

$$= 1$$

Let the member BC be the redundant.

Replace BC and apply a force (R)  $\rightarrow$  compressive

$$\frac{\partial U}{\partial R} = 0$$

$$U = \sum \frac{F^2 L}{2AE}$$

$$\frac{\partial U}{\partial R} = \sum \frac{L}{AE} \cdot F \cdot \frac{\partial F}{\partial R} = 0$$

Member	Length (m)	Area (A)	Force F	$\frac{\partial F}{\partial R}$	$\frac{L}{AE} \cdot F \cdot \frac{\partial F}{\partial R} \cdot aE$	Actual Force
AB	1.5	a	$0.6R + 0.75W$	0.6	$0.4R + 67.5W$	1.088W (T)
AC	2.0	a	$0.8R$	0.8	$1.28R$	0.451W (T)
AD	2.5	a	$1.25W - R$	-1	$-312.5W + 250R$	0.686W (T)
BC	2.5	a	$-R$	-1	$250R$	0.564W (C)
BD	2.0	a	$-(W - 0.8R)$	0.8	$-160W + 128R$	0.549W (C)
BE	2.5	a	$1.25W$	0	0	1.25W (T)
CD	1.5	2a	$-(1.25W - 0.6R)$	0.6	$-67.5W + 87R$	1.162W (C)
DE	1.5	2a	$0.75W$	0	0	0.75W (C)
					$837R - 472.5W$	





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Joint E :



$$\sum V = 0,$$

$$F_{BE} \sin \theta = W$$

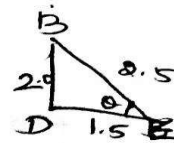
$$F_{BE} = \frac{W}{\sin \theta} = \frac{W}{4/5} = 1.25W \quad (\text{or}) \quad \frac{5W}{4}$$

$$\sum H = 0,$$

$$F_{DE} = F_{BE} \cos \theta$$

$$= 1.25W \times 3/5$$

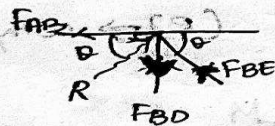
$$F_{DE} = 0.75W \quad (\text{compressive})$$



$$\sin \theta = \frac{2.0}{2.5}$$

$$\cos \theta = \frac{1.5}{2.5}$$

Joint B :



$$\sum V = 0,$$

$$F_{BD} - R \sin \theta - F_{BE} \sin \theta = 0$$

$$F_{BD} = \left( \frac{5}{4}W \times \frac{4}{5} \right) + \left( R \times \frac{4}{5} \right)$$

$$F_{BD} = W + 0.8R \quad (\text{compressive})$$

$$\sum H = 0,$$

$$F_{AB} - F_{BE} \cos \theta - R \cos \theta = 0$$

$$F_{AB} = \frac{3}{5}R + \frac{5}{4}W \times \frac{3}{5}$$

$$F_{AB} = (0.6R + 0.75W)$$

Tension.



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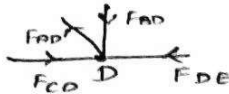
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Joint D :



$$\sum V = 0, \quad F_{AD} \sin \theta = F_{BD}$$

$$F_{AD} = (W - 0.8R) \frac{5}{4}$$

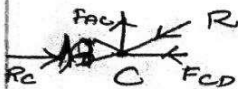
$$F_{AD} = 1.25W - R \quad (\text{Tension})$$

$$\sum H = 0, \quad F_{CD} - F_{DE} - F_{AD} \cos \theta = 0$$

$$F_{CD} = (1.25W - R) \frac{3}{5} + 0.75W$$

$$F_{CD} = 1.5W - 0.6R \quad (\text{Compressive})$$

Joint C :



$$\sum V = 0, \quad F_{AC} - R \sin \theta = 0$$

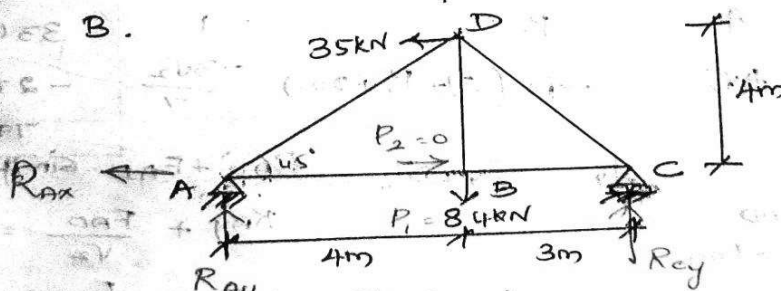
$$F_{AC} = R \times \frac{4}{5} = 0.8R$$

(Tension)

$$\frac{\partial U}{\partial R} = 0, \quad 837R - 472.5W = 0$$

$$R = 0.564W$$

12) Use Castigliano's second theorem to determine the horizontal and vertical components of the deflection at joint B.



$$EA = \text{constant}, \quad E = 200 \text{ Gpa}$$

$$A = 1200 \text{ mm}^2$$

$$m = 5, \quad R = 3$$

$$J = 4$$

Soln :

Static Indeterminacy :

$$D_s = (m + R) - 2J$$

$$= (5 + 3) - (2 \times 4) = 0$$



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Soln :

Unknown Reactions :

$$\sum M_A = 0, \quad R_{cy}$$

$$R_{cy} \times 7 + 35(4) - (P_1 \times 4) = 0$$

$$R_{cy} = \frac{4P_1}{7} - 20$$

$$\sum F_v = 0,$$

$$R_{cy} + R_{Ay} - P_1 = 0$$

$$R_{Ay} = P_1 - R_{cy}$$

$$= P_1 - \left( \frac{4P_1}{7} - 20 \right)$$

$$R_{Ay} = \frac{3P_1}{7} + 20$$

$$\sum H = 0,$$

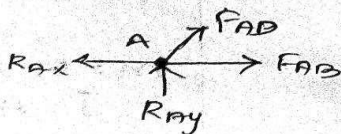
$$-R_{Ax} + P_2 - 35 = 0$$

$$R_{Ax} = P_2 - 35$$

Deflection at B :

Member	Length (m)	Force	$\frac{\partial F}{\partial P_1}$	$\frac{FL}{AE} \frac{\partial F}{\partial P_1} =$
AB	4	$P_2 + 3/4 P_1 - 15$	3/4	$36/AE$
BC	3	$3/4 P_1 - 15$	3/4	$27/AE$
CD	5	$5/4 P_1 - 25$	5/4	$125/AE$
BD	4	$P_1$	1	$336/AE$
AD	$4\sqrt{2}$	$-\sqrt{2} (3/4 P_1 + 20)$	$-\frac{3\sqrt{2}}{4}$	$-271.53/AE$
				$795.53/AE$

Joint A :



$$\sum v = 0,$$

$$R_{Ay} + F_{AD} \sin 45^\circ = 0$$

$$R_{Ay} + \frac{F_{AD}}{\sqrt{2}} = 0$$

$$\frac{F_{AD}}{\sqrt{2}} = -R_{Ay}$$

$$F_{AD} = -R_{Ay} \sqrt{2}$$

$$F_{AD} = -\sqrt{2} \left( \frac{3P_1}{7} + 20 \right)$$





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$$\sum H = 0, \quad P_2 - 35 = R_{Ax} \quad \text{---}$$

$$-R_{Ax} + (F_{AD} \times \frac{1}{\sqrt{2}}) + F_{AB} = 0$$

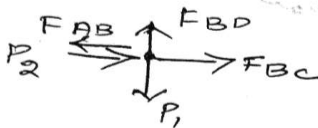
$$F_{AB} = R_{Ax} - F_{AD} \times \frac{1}{\sqrt{2}}$$

$$= (P_2 - 35) - (-\sqrt{2})(\frac{3}{4}P_1 + 20) \times \frac{1}{\sqrt{2}}$$

$$= P_2 - 35 + \frac{3}{4}P_1 + 20$$

$$= P_2 + \frac{3}{4}P_1 - 15$$

Joint B:



$$\sum V = 0,$$

$$F_{BD} - P_1 = 0$$

$$F_{BD} = P_1$$

$$\sum H = 0,$$

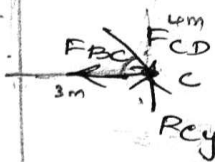
$$-F_{BC} + P_2 - F_{AB} = 0$$

$$F_{BC} = F_{AB} - P_2$$

$$= (P_2 + \frac{3}{4}P_1 - 15) - P_2$$

$$= \frac{3}{4}P_1 - 15$$

Joint C:



$$\sum V = 0,$$

$$R_{Cy} - F_{CD} \sin \theta = 0$$

$$\sin \theta = \frac{OP}{OQ} = \frac{4}{5}$$

$$R_{Cy} - F_{CD} \times \frac{4}{5} = 0$$

$$\frac{4}{5} F_{CD} = R_{Cy}$$

$$F_{CD} = \frac{5}{4} \times (\frac{3}{4}P_1 - 20)$$

$$= \frac{5}{4}P_1 - 25$$

from table,

$$\Delta V_B = \frac{795.33 \times 1000 \times 1000}{1200 \times 200 \times 10^3} \quad \text{N.mm}$$

$$= 3.315 \text{ mm } (\downarrow)$$



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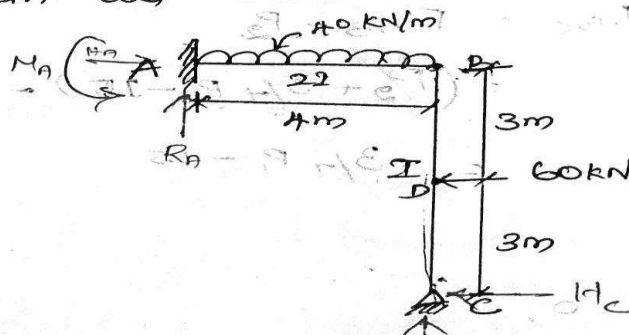
Member	Length (m)	Force	$\frac{\partial P}{\partial P_2}$
AB	4	$P_2 + 3/4 P_1 - 15$	1
BC	3	$3/4 P_1 - 15$	0
CD	5	$5/4 P_1 - 25$	0
BD	4	$P_1$	0
AD	$4\sqrt{2}$	$-\sqrt{2}(3/4 P_1 + 20)$	0

$$\Delta_{HB} = \frac{84 \times 1000 \times 1000}{1200 \times 200 \times 10^3} \frac{\text{N} \cdot \text{mm}}{\text{mm}^2 \times \text{N}/\text{mm}^2}$$

$$\Delta_{HB} = 0.35 \text{ mm } (\rightarrow)$$

$$\Delta_B = \sqrt{\Delta_H^2 + \Delta_V^2}$$

13. Analyse the frame as shown in figure by Castigliano's theorem and sketch the bending moment diagram.



$$3m + R - 3j$$

$$3(3) + 5$$

Soln :

Static Indeterminacy :

$$M = 3, \quad R = 5$$

$$J = 4$$

$$D_s = 3m + R - 3j$$

$$= (3 \times 3) + 5 - (3 \times 4)$$

$$= 14 - 12$$

$$= 2$$

$R_c$  and  $H_c$  are considered as redundant

Remove the support to make given structure a determinate.



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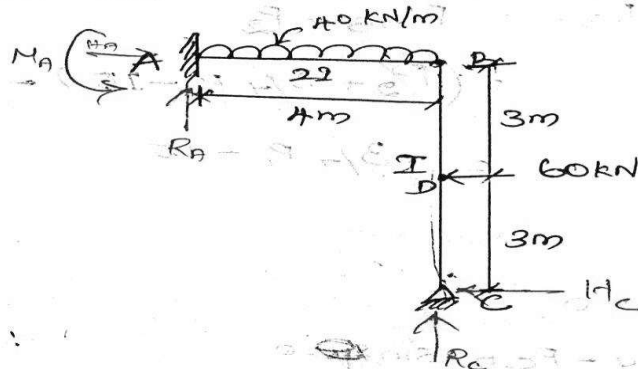
Member	Length (m)	Force	$\frac{\partial P}{\partial P_2}$
AB	4	$P_2 + 3/4 P_1 - 15$	1
BC	3	$3/4 P_1 - 15$	0
CD	5	$5/4 P_1 - 25$	0
BD	4	$P_1$	0
AD	$4\sqrt{2}$	$-\sqrt{2}(3/4 P_1 + 20)$	0

$$\Delta_{HB} = \frac{84 \times 1000 \times 1000}{1200 \times 200 \times 10^9} \frac{\text{N} \cdot \text{mm}}{\text{mm}^2 \times \text{N}/\text{mm}^2}$$

$$\Delta_{HB} = 0.35 \text{ mm (} \rightarrow \text{)}$$

$$\Delta_B = \sqrt{\Delta_H^2 + \Delta_V^2}$$

13. Analyse the frame as shown in figure by Castigliano's theorem and sketch the bending moment diagram.



Sol :

Static Indeterminacy :

$$D_s = 3m + R - 3j$$

$$= (3 \times 3) + 5 - (3 \times 4)$$

$$= 14 - 12$$

$$= 2$$

$$m = 3, \quad R = 5$$

$$j = 4$$

$R_c$  and  $H_c$  are considered as redundant. Remove the support to make given structure a determinate.



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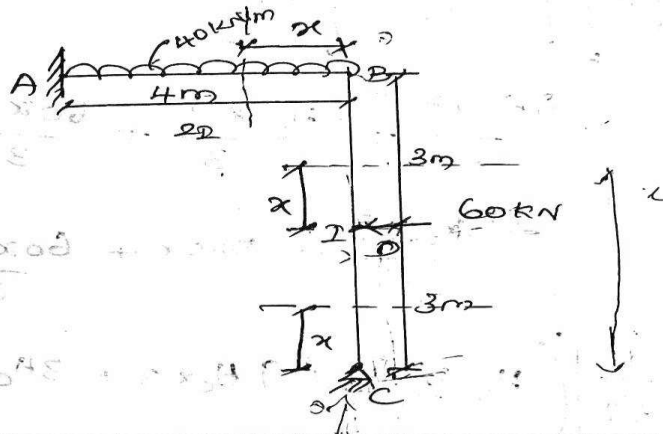


Redundant Calculation

$$U = \int \frac{M^2}{2EI} \cdot dx$$

$$\frac{\partial U}{\partial H_c} = 0 \Rightarrow \frac{\partial U}{\partial H_c} = \frac{1}{EI} \int M \frac{\partial M}{\partial H_c} \cdot dx = 0 \rightarrow \textcircled{1}$$

$$\frac{\partial U}{\partial R_c} = 0 \Rightarrow \frac{\partial U}{\partial R_c} = \frac{1}{EI} \int M \cdot \frac{\partial M}{\partial R_c} \cdot dx = 0 \rightarrow \textcircled{2}$$



Segment	Origin	Limit	$M_x$	$\frac{\partial M}{\partial H_c}$	$\frac{\partial M}{\partial R_c}$	I
CD	C	0 to 3	$-H_c \cdot x$	$-x$	0	I
DB	D	0 to 3	$-H_c(3+x) - 60x$	$-(3+x)$	0	I
BA	B	0 to 4	$(-H_c \cdot 4) + (R_c \cdot x) - (60 \cdot 3) - 40 \frac{x^2}{2}$	$-4$	$x$	2I

$$\textcircled{1} \Rightarrow \frac{\partial U}{\partial H_c} = \frac{1}{EI} \int_0^3 (-H_c x)(-x) dx + \frac{1}{EI} \int_0^3 (-H_c(3+x) - 60x)(-(3+x)) dx$$

$$0 = \frac{1}{EI} \int_0^4 (-6H_c + R_c x - 180 - 20x^2)(-6) dx$$

$$\frac{1}{EI} \left[ \int_0^3 H_c x^2 \cdot dx + \int_0^3 (-H_c \cdot 3 - H_c x - 60x)x \cdot (-3-x) \cdot dx + \int_0^4 18H_c - 3R_c x + 540 + 60x^2 \cdot dx \right]$$



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$$\begin{aligned}\frac{\partial U}{\partial H_c} &= \frac{1}{EI} \left[ \int_0^3 H_c x^2 \cdot dx + \int_0^3 (9H_c + 3H_c x + 180x - 3H_c x + H_c x^2 + 60x^2) \right. \\ &\quad \left. + \int_0^4 (18H_c - 3R_c x + 540 + 60x^2) \cdot dx \right] = 0 \\ &= \frac{1}{EI} \left[ \left[ H_c \frac{x^3}{3} \right]_0^3 + \left[ 9H_c x + 3H_c \frac{x^2}{2} + \frac{180x^2}{2} \right. \right. \\ &\quad \left. \left. - 3H_c \frac{x^2}{2} + H_c \frac{x^3}{3} + \frac{60x^3}{3} \right]_0^3 + \left[ 18H_c x - \frac{3R_c x^2}{2} + 540x + \frac{60x^3}{3} \right]_0^4 \right] = 0 \\ &= \frac{1}{EI} \left[ \left( H_c \frac{3^3}{3} \right) + \left( 9H_c \times 3 + 3H_c \times \frac{3^2}{2} + \frac{180 \times 3^2}{2} \right. \right. \\ &\quad \left. \left. - 3H_c \times \frac{3^2}{2} + H_c \times \frac{3^3}{3} + \frac{60 \times 3^3}{3} \right) + \left( 18H_c \times 4 - \frac{3R_c \times 4^2}{2} \right. \right. \\ &\quad \left. \left. + (540 \times 4) + \frac{60 \times 4^3}{3} \right) \right] = 0\end{aligned}$$

$$144 H_c - 24 R_c = -4790 \rightarrow \textcircled{3}$$

$$\frac{\partial U}{\partial R_c} = 0 \Rightarrow \int_0^4 (-H_c \times 6 + R_c x - 180 + 20x^2) (x) dx = 0$$

$$\frac{1}{2EI} \int_0^4 (-H_c \times 6x + R_c x^2 - 180x + 20x^3) dx = 0$$

$$\frac{1}{2EI} \left[ -H_c \times \frac{6x^2}{2} + R_c \frac{x^3}{3} - \frac{180x^2}{2} + \frac{20x^4}{4} \right]_0^4 = 0$$

$$\frac{1}{2EI} \left[ -H_c \times \frac{6 \times 4^2}{2} + R_c \times \frac{4^3}{3} - \frac{180 \times 4^2}{2} + \frac{20 \times 4^4}{4} \right] = 0$$



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$$-24H_c + 10.65 R_c = 1360 \rightarrow (4)$$

By solving (3) & (4)

$$H_c = -19.18 \text{ kN } (\rightarrow)$$

$$R_c = 84.46 \text{ kN}$$

For CD :

$$M_x = -H_c \times x \quad x=0, M_c = 0$$

$$M_D = -(-19.18 \times 3) = 57.54 \text{ kNm}$$

for DB :

$$M_x = -H_c(3+x) - 60x$$

$$x=0, M_D = -(-19.18)(3+0) = 57.54 \text{ kNm}$$

$$x=3\text{m}, M_B = -(-19.18)(3+3) - (60 \times 3) \\ = -64.92 \text{ kNm}$$

For BA :

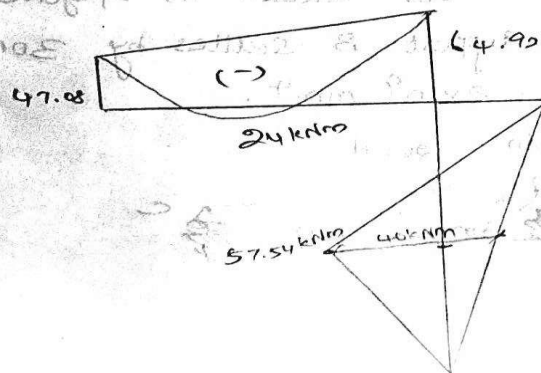
$$M_x = (-H_c \times 6) + (R_c \times x) - (60 \times 3) - \frac{40x^2}{2}$$

$$x=0, M_B = -(-19.18) \times 6 + 84.46 \times 0 - 180 - \frac{40 \times 0}{2}$$

$$M_B = -64.92 \text{ kNm}$$

$$x=4, M_A = -(-19.18 \times 6) + (84.46 \times 4) - 180 - \frac{40 \times 4^2}{2}$$

$$= -47.08 \text{ kNm}$$



$$\frac{Wl^2}{8} = \frac{10 \times 4^2}{8}$$

$$= 20 \text{ kNm}$$

$$\frac{Wl}{4} = \frac{60 \times 6}{4} = 90 \text{ kNm}$$



# SNS COLLEGE OF TECHNOLOGY

(An Autonomous Institution)

Approved by AICTE, New Delhi, Affiliated to Anna University, Chennai

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Accredited by NBA (B.E - CSE, EEE, ECE, Mech & B.Tech.IT)

COIMBATORE-641 035, TAMIL NADU

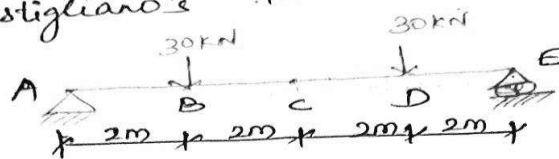
ACADEMIC YEAR 2024-25

Department of Civil Engineering ( SA I)



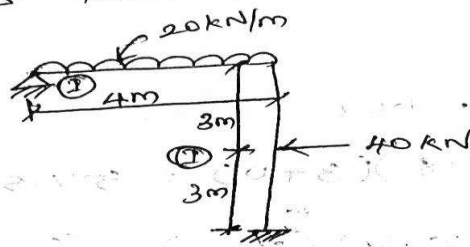
## Assignment - 1

1. Calculate the vertical displacement of point C using Castigliano's theorem.



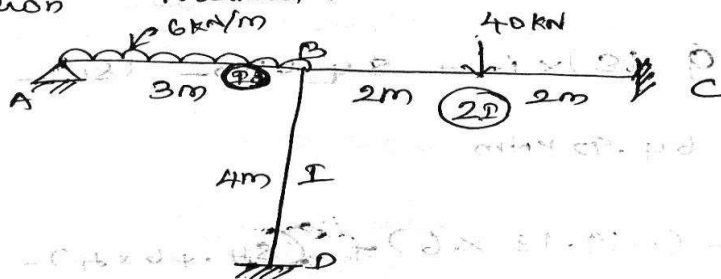
$$I = 600 \times 10^6 \text{ mm}^4$$
$$E = 200 \text{ Gpa}$$

2. Analyse the frame as shown in figure by Castigliano's theorem.

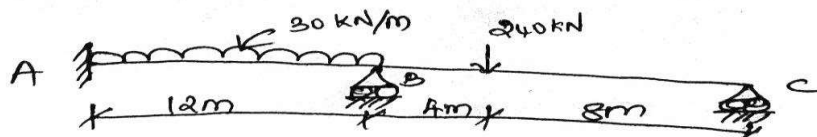


## Assignment - 2

1. Analyse the frame as shown in figure by slope deflection method.



2. Analyse the beam as shown in figure by slope deflection method. Support B settles by 30mm. Take  $E = 2 \times 10^5 \text{ N/mm}^2$ ,  $I = 2 \times 10^9 \text{ mm}^4$ .



M. S. S.