

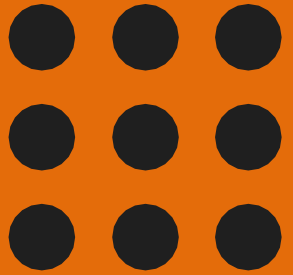


SNS COLLEGE OF TECHNOLOGY

Coimbatore-35

(An Autonomous Institution)

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DEPARTMENT OF AEOSPACE ENGINEERING

FINITE ELEMENT ANALYSIS

UNIT III TWO DIMENSIONAL PROBLEMS

TOPIC – Constant Strain Triangular element



SNS *Design Thinkers*

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Formulae used

For constant strain triangle (CST) element,

$$\text{Shape function, } N_1 + N_2 + N_3 = 1$$

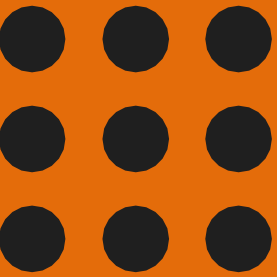
$$\text{Co-ordinate, } x = N_1 x_1 + N_2 x_2 + N_3 x_3$$

$$\text{Co-ordinate, } y = N_1 y_1 + N_2 y_2 + N_3 y_3$$

or

$$\text{Co-ordinate, } x = (x_1 - x_3) N_1 + (x_2 - x_3) N_2 + x_3$$

$$\text{Co-ordinate, } y = (y_1 - y_3) N_1 + (y_2 - y_3) N_2 + y_3$$





Area of the triangular element, $A = \frac{1}{2} \begin{vmatrix} 1 & x_1 & y_1 \\ 1 & x_2 & y_2 \\ 1 & x_3 & y_3 \end{vmatrix}$

Strain-Displacement matrix for CST element is,

$$[B] = \frac{1}{2A} \begin{bmatrix} q_1 & 0 & q_2 & 0 & q_3 & 0 \\ 0 & r_1 & 0 & r_2 & 0 & r_3 \\ r_1 & q_1 & r_2 & q_2 & r_3 & q_3 \end{bmatrix}$$

where, $q_1 = y_2 - y_3; \quad q_2 = y_3 - y_1; \quad q_3 = y_1 - y_2$
 $r_1 = x_3 - x_2; \quad r_2 = x_1 - x_3; \quad r_3 = x_2 - x_1$



Stress-Strain relationship matrix for plane stress problem,

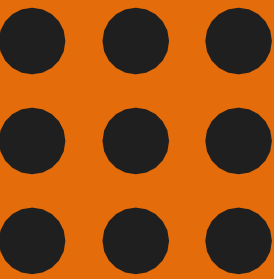
$$[D] = \frac{E}{1-\nu^2} \begin{bmatrix} 1 & \nu & 0 \\ \nu & 1 & 0 \\ 0 & 0 & \frac{1-\nu}{2} \end{bmatrix}$$

where, $\nu \rightarrow$ Poisson's ratio

$E \rightarrow$ Young's modulus

Stress-Strain relationship matrix for plane strain problem,

$$[D] = \frac{E}{(1+\nu)(1-2\nu)} \begin{bmatrix} 1-\nu & \nu & 0 \\ \nu & 1-\nu & 0 \\ 0 & 0 & \frac{1-2\nu}{2} \end{bmatrix}$$





Element stiffness matrix for CST element,

$$[K] = [B]^T [D] [B] A t$$

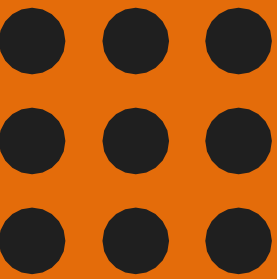
Element stress, $\{ \sigma \} = [D] [B] \{ u \}$

$$\Rightarrow \begin{Bmatrix} \sigma_x \\ \sigma_y \\ \tau_{xy} \end{Bmatrix} = [D] [B] \begin{Bmatrix} u_1 \\ v_1 \\ u_2 \\ v_2 \\ u_3 \\ v_3 \end{Bmatrix}$$

where, $\sigma_x, \sigma_y \rightarrow$ Normal stresses

$\tau_{xy} \rightarrow$ Shear stress

$u, v \rightarrow$ Nodal displacements



Maximum normal stress, $\sigma_{max} = \sigma_1 = \frac{\sigma_x + \sigma_y}{2} + \sqrt{\left(\frac{\sigma_x - \sigma_y}{2}\right)^2 + \tau_{xy}^2}$

Minimum normal stress, $\sigma_{min} = \sigma_2 = \frac{\sigma_x + \sigma_y}{2} - \sqrt{\left(\frac{\sigma_x - \sigma_y}{2}\right)^2 + \tau_{xy}^2}$

Principal angle, $\tan 2\theta_p = \frac{2\tau_{xy}}{\sigma_x - \sigma_y}$

Element strain, $\{e\} = [B] \{u\} = [B] \begin{Bmatrix} u_1 \\ v_1 \\ u_2 \\ v_2 \\ u_3 \\ v_3 \end{Bmatrix}$





Temperature effects

$$\left. \begin{array}{l} \text{Initial strain, } \{ e_0 \} \\ \text{(For plane stress problems)} \end{array} \right\} = \left\{ \begin{array}{l} \alpha \Delta T \\ \alpha \Delta T \\ 0 \end{array} \right\}$$

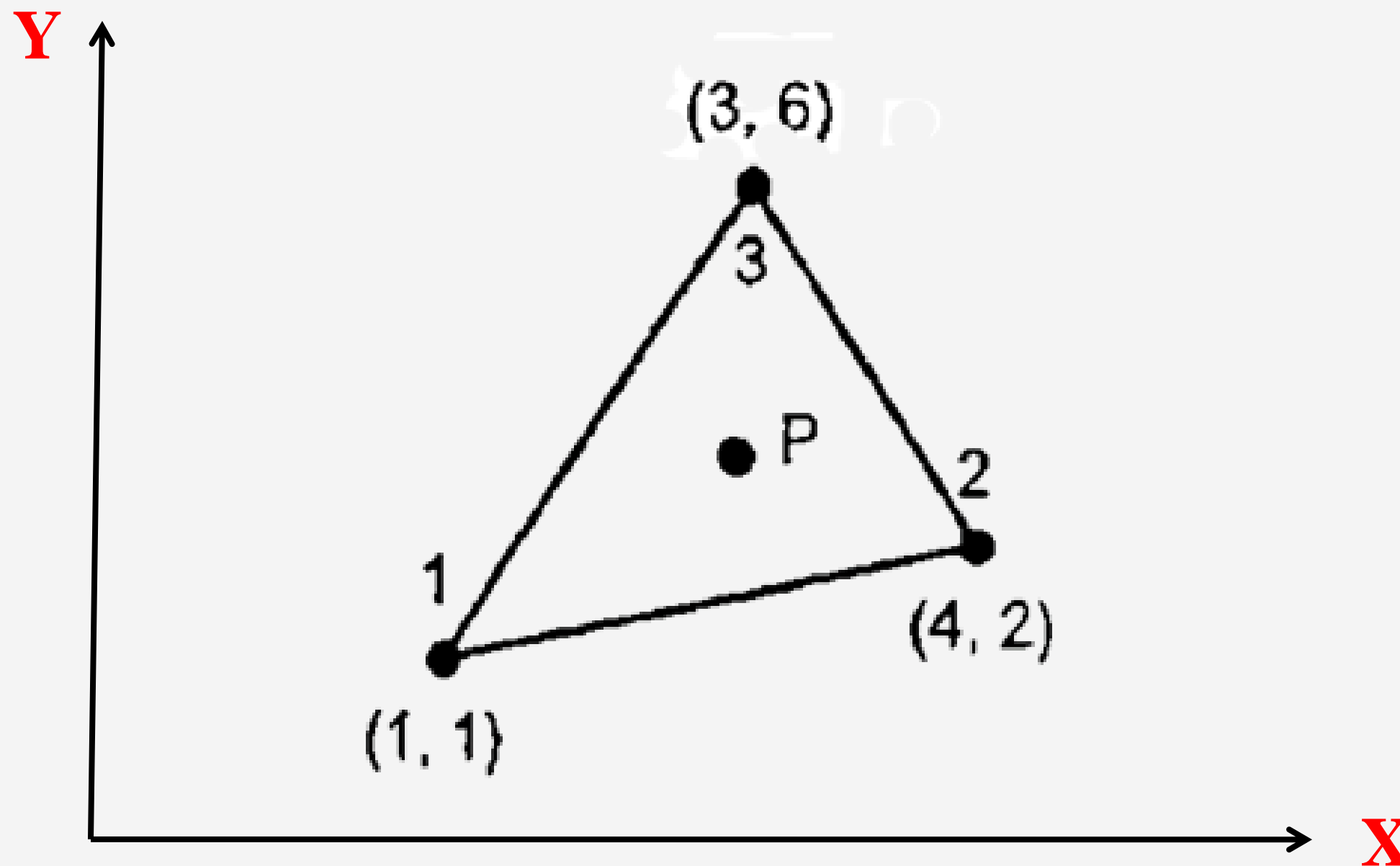
$$\left. \begin{array}{l} \text{Initial strain, } \{ e_0 \} \\ \text{(For plane strain problems)} \end{array} \right\} = (1 + \nu) \left\{ \begin{array}{l} \alpha \Delta T \\ \alpha \Delta T \\ 0 \end{array} \right\}$$

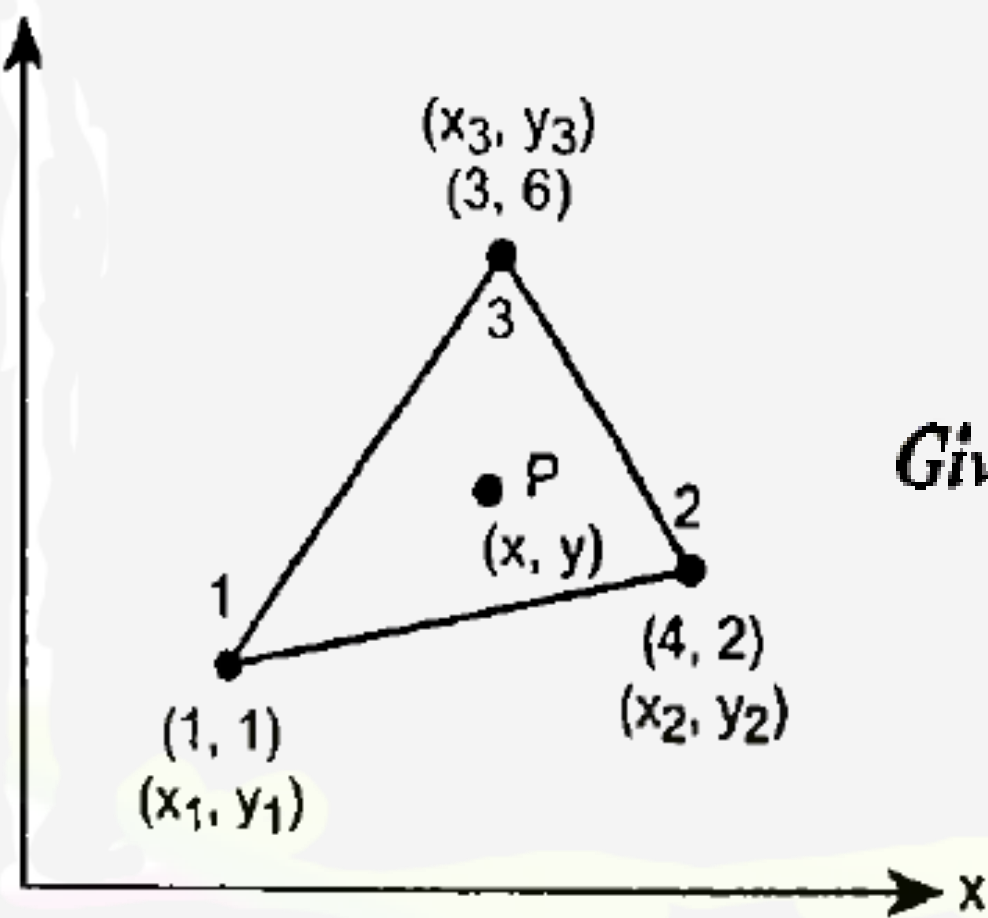
where, $\alpha \rightarrow$ Coefficient of thermal expansion
 $\nu \rightarrow$ Poisson's ratio

$$\text{Element temperature force, } \{ F \} = [B]^T [D] \{ e_0 \} t A$$



Determine the x and y co-ordinates of point P for the triangular element shown in Fig. . The shape functions N_1 and N_2 are 0.2 and 0.3 respectively.





Given:

$x_1 = 1$	$y_1 = 1$	$x_2 = 4$	$y_2 = 2$
$x_3 = 3$	$y_3 = 6$	$N_1 = 0.2$	$N_2 = 0.3$

To find: x and y co-ordinates of point P.

Solution: We know that,

Sum of shape function is equal to one.

$$\Rightarrow N_1 + N_2 + N_3 = 1$$

$$0.2 + 0.3 + N_3 = 1$$

$N_3 = 0.5$



x co-ordinate at point P is,

$$x = N_1 x_1 + N_2 x_2 + N_3 x_3$$

$$= 0.2 \times (1) + 0.3 \times (4) + 0.5 \times (3)$$

$$x = 2.9$$

y co-ordinate at point P is,

$$y = N_1 y_1 + N_2 y_2 + N_3 y_3$$

$$= 0.2 \times (1) + 0.3 \times (2) + 0.5 \times (6)$$

$$y = 3.8$$

Thank you

