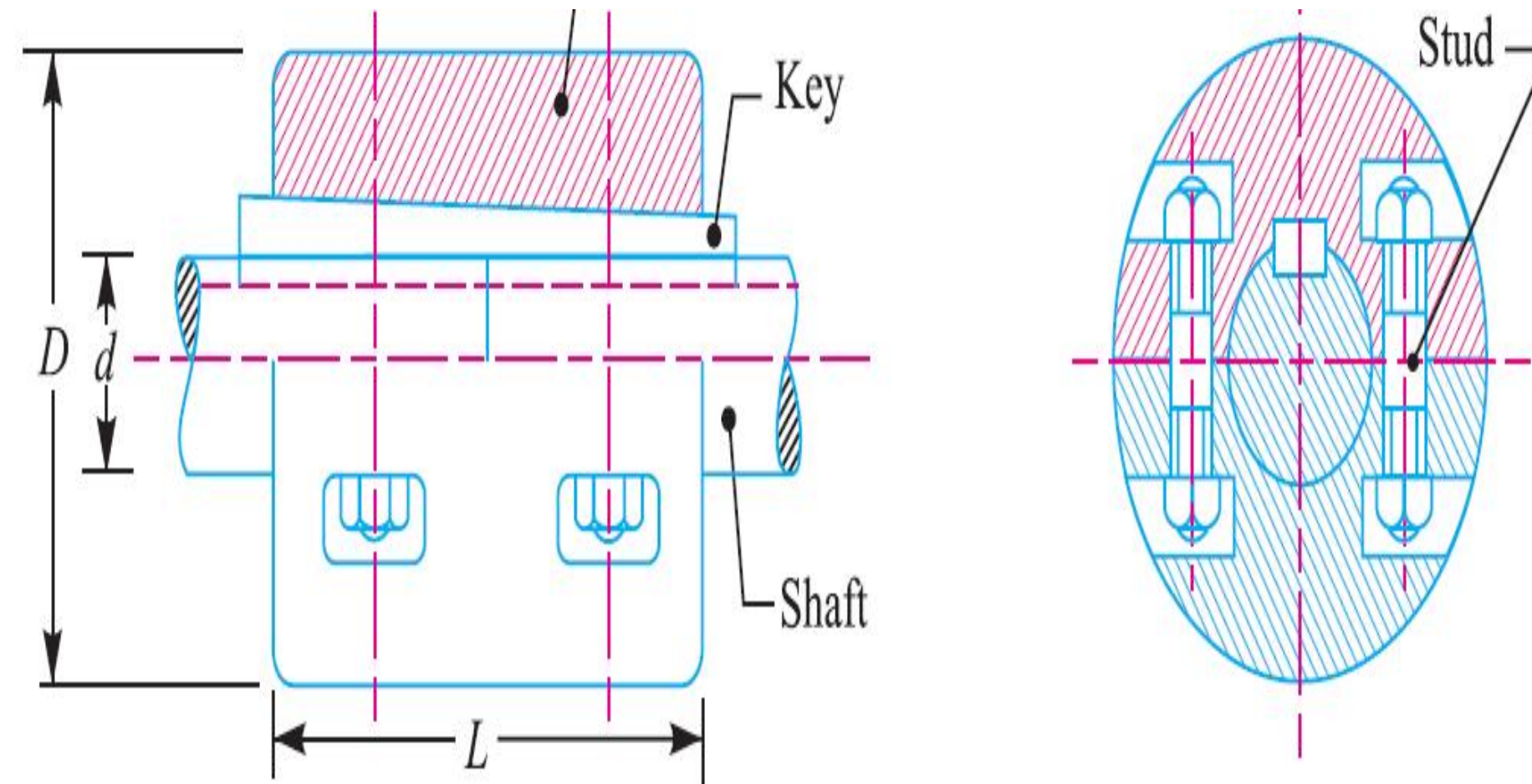




Design of Split -Muff Coupling

Source:Text book by RS Khurmi



Split Muff Coupling



Design of Rigid Flange Coupling



- (d): Diameter of shaft
- (D): Diameter of Hub
- (w): Width of key
- (h): Height of key
- (l): Length of key
- (L): Length of hub
- (D): Outside diameter
- (d_c): Core diameter
- (d_b): Nominal diameter



Design of Shaft

Parameter to be calculated

Stress Induced

Equation

Diameter of shaft (d):

Maximum shear stress
(τ_{max})

$$\tau_{max} = \frac{16T}{\pi d^3}$$

Source:Indiamart.com



Shaft



Design of Key

Parameter to be calculated

Stress Induced

Equation

Key dimensions:

- i. Width of key (w)
- ii. Height of key (h)
- iii. Length of key (l)

By proportion

By proportion

- By proportion
- Direct shear stress (τ_d)
- Crushing stress (σ_c)

$$w = d/4$$

$$h = (2/3)w$$

for rectangular key

$$h = w$$

for square key

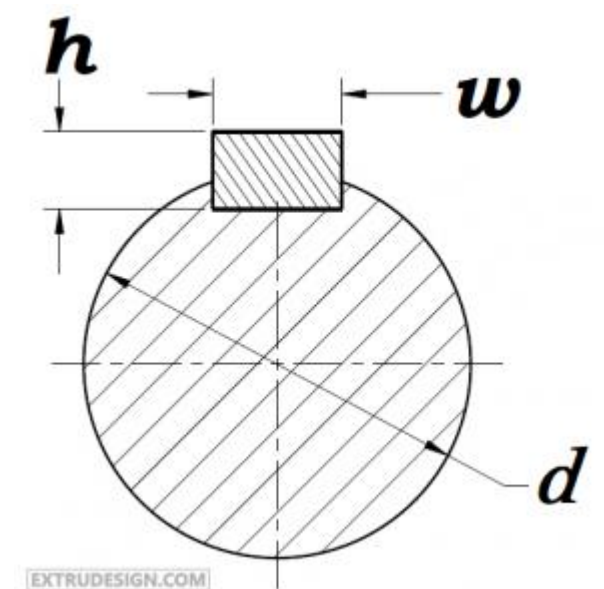
$$l = L/2 = 1.75d$$

$$\tau_d = \frac{2T}{dwl}$$
$$\sigma_c = \frac{4T}{dlh}$$

Source: Researchgate.com



Rectangular Key





Design of Hub



Parameter to be calculated

Stress Induced

Equation

Dimensions of hub:

- i. Length of hub (l)**
- ii. Outside diameter (D)**
- iii. Stress in hub**

By proportion
By proportion
Torsional shear stress (τ_h)

l is taken as length of key
 $D=2d$

$$\tau_h = \frac{16T}{\pi D^3(1 - K^4)}$$

SourceIndianmart.com



Split Muff



Design of Sleeve Bolt



Parameter to be calculated

Stress Induced

Equation

Source:indianmart.com

Dimensions of clamping bolts:

i. Core diameter(d_c)

Tensile stress(σ_t)

i. Nominal diameter (d_b)

By proportion

$$d_c = \sqrt{\frac{16T}{\pi^2 \mu N d \sigma_t}}$$

$$d_b = d_c / 0.84$$



Bolts and Nuts



References



- ❑ <https://www.machinedesign.com/archive/article/21817862/best-sites-on-the-web>
- ❑ <https://www.machinedesignonline.com>
- ❑ <https://www.quora.com/What-are-the-websites-a-machine-design-lover-should-visit>
- ❑ <https://imechanica.org/files/theories%20of%20failure.pdf>

Thank You