



1. A vertical plate 30 cm high and 1 m wide and maintained at a uniform temperature of  $120^{\circ}\text{C}$  is exposed to quiescent air at  $30^{\circ}\text{C}$ . Calculate the average heat transfer coefficient and the total heat transfer rate from the plate to air.

Sol: Given:  $L = 30\text{cm}$ ;  $W = 1\text{m}$ .  $T_w = 120^{\circ}\text{C}$   $T_a = 30^{\circ}\text{C}$

To find: (a)  $h = ?$  (b)  $Q = ?$

$$T_{\text{film}} = \frac{T_w + T_a}{2} = \frac{120 + 30}{2} = 75^{\circ}\text{C}$$

Properties of air @  $75^{\circ}\text{C}$  from data handbook  
 $\beta = 1.015 \text{ K}^{-1}$ ;  $\mu = 20.9 \times 10^{-6} \text{ Pa}\cdot\text{s}$   $\nu = 20.6 \times 10^{-6} \text{ m}^2/\text{s}$ .

$Pr = 0.693$   $C_p = 1.009 \text{ kJ/kg}\cdot\text{K}$   $K = 0.03025 \text{ W/m}\cdot\text{K}$ .

$$\beta = \frac{1}{(T_{\text{film}} + 273)} = \frac{1}{(75 + 273)} = 2.87 \times 10^{-3} / ^{\circ}\text{K}$$

$$Gr = \frac{\beta g \Delta T L^3}{\nu^2} = \frac{2.87 \times 10^{-3} \times 9.81 \times (120 - 30) \times 0.3^3}{(20.6 \times 10^{-6})^2}$$

$$Gr \approx 1.6 \times 10^8$$

$$Ra = Gr Pr = 1.6 \times 10^8 \times 0.693 = 1.16 \times 10^8 < 10^9$$

The flow is laminar.

$$Nu_x = 0.508 Pr^{0.5} (0.952 + Pr)^{-0.25} \cdot Gr_x^{0.25} \\ = 42.$$



$$Nu_{avg} = \frac{4}{3} Nu_x = \frac{4}{3} \times 42 = 56$$

$$Nu = \frac{hL}{k} \rightarrow h_{avg} = \frac{Nu_{avg} k}{L} = \frac{56 \times 0.03025}{0.3}$$

⊙  $h_{avg} = 5.61 \text{ W/m}^2\text{-K}$ .

⊙  $Q = h_{avg} A (T_w - T_m)$   
 $= 5.61 \times (0.2 \times 1) (120 - 30)$

$Q = 151 \text{ W}$ .

Case 1:-

By calculating the  $Nu \rightarrow h \rightarrow Q$  using Churchill & Chu correlation given in pg.136 of data book.

$$\overline{Nu} = \left[ 0.825 + \frac{0.287 (GrPr)^{0.167}}{\left[ 1 + \left( \frac{0.492}{Pr} \right)^{0.5625} \right]^{0.276}} \right]^2$$

$\overline{Nu} = 63 \rightarrow h = 6.38 \text{ W/m}^2\text{K} \rightarrow Q = 172 \text{ W}$ .

Case 2: With radiation:

$$Q_{rad} = \sigma A (T_w^4 - T_m^4) = 5.67 \times 10^{-8} \times (0.2 \times 1) (493^4 - 303^4)$$

$= 262.28 \text{ W}$ .

$$Q_{total} = Q_{conv} + Q_{rad} = 172 + 262 = 435 \text{ W}$$



2. A flat electrical heater of 0.4 m X 0.4 m size is placed vertically in still air at 20°C. The heat generated is 1200W/m<sup>2</sup>. Determine the value of convective heat transfer coefficient and the average plate temperature.

Sol:  $A = 0.4 \times 0.4 \text{ m}^2$ ;  $L = 0.4 \text{ m}$ ;  $B = 0.4 \text{ m}$ .  $q_f = 1200 \text{ W/m}^2$ .

$$T_m = 20^\circ\text{C}$$

To find  $h$ ,  $T_w = ?$  Iteration 1:

Assuming an average value of  $h = 6 \text{ W/m}^2\text{K}$ .

$$\therefore q_f = h\Delta T \rightarrow \Delta T = \frac{q_f}{h} = \frac{1200}{6} = 200^\circ\text{C}$$

$$T_w - T_m = 200 \rightarrow T_w = 200 + 20 = 220^\circ\text{C}$$

$$\therefore T_f = \frac{220 + 20}{2} = 120^\circ\text{C} \quad \beta = \frac{1}{(120 + 273)} = 2.54 \times 10^{-3} / ^\circ\text{C}$$

Property values of air @ 120°C

$$\rho = 0.876 \text{ kg/m}^3; \nu = 26.625 \times 10^{-6} \text{ m}^2/\text{s}; Pr = 0.685$$

$$k = 0.03118 \text{ W/mK}; C_p = 1011 \text{ J/kgK}$$

$$Gr = \frac{g\beta\Delta TL^3}{\nu^2} = \frac{9.81 \times 2.54 \times 10^{-3} \times 1200 \times 0.4^3}{(26.625 \times 10^{-6})^2}$$

$$Gr = 2.69 \times 10^9$$

$$Ra_L = Gr \cdot Pr = 2.69 \times 10^9 \times 0.685 = 1.85 \times 10^9 > 10^9 \text{ (Turbulent)}$$

$$Nu_{x,c} = 0.17 (Gr_L Pr)^{0.25} = 0.17 \times (1.85 \times 10^9)^{0.25}$$

$$Nu_{x,c} = 35.26$$

$$\overline{Nu}_L = \frac{5}{4} Nu_{x,c} = \frac{5}{4} \times 35.26 = 44.075$$



$$\bar{h}_L = \frac{Nu_L \cdot k}{L} = \frac{44.075 \times 0.01138}{0.4} = 3.68 \text{ W/m}^2\text{K}$$

Iteration-2: Assume  $h = 4.5 \text{ W/m}^2\text{K}$ .

$$q = h \Delta T \rightarrow \Delta T = \frac{q}{h} = \frac{1200}{4.5} = 267$$

$$T_w = 267 + 20 = 287^\circ\text{C} \rightarrow T_f = \frac{287 + 20}{2} = 153.5 = 154^\circ\text{C}$$

Properties of air @  $150^\circ\text{C}$

$$\rho = 0.83 \text{ kg/m}^3 \quad \nu = 28.5 \times 10^{-6} \text{ m}^2/\text{s} \quad Pr = 0.681 \quad k = 0.0352 \text{ W/mK}$$

$$Gr = \frac{2.24 \times 10^{-3} \times 9.81 \times 267 \times 0.4^3}{(28.5 \times 10^{-6})^2} = 0.484 \times 10^9$$

$$Re = Gr Pr = 0.484 \times 10^9 \times 0.681 = 0.33 \times 10^9 < 10^9 \text{ (Laminar)}$$

$$Nu_x = 0.6 (Gr Pr)^{0.2} = 0.6 (0.33 \times 10^9)^{0.2} = 30.34$$

$$\bar{Nu}_L = \frac{4}{3} \times 30.34 = 40.45$$

$$\bar{h}_L = \frac{\bar{Nu}_L \cdot k}{L} = \frac{40.45 \times 0.0352}{0.4} = 3.6 \text{ W/m}^2\text{K}$$

Iteration 3: Assume  $h = 4 \text{ W/m}^2\text{K}$ .

$$\Delta T = \frac{q}{h} = \frac{1200}{4} = 300^\circ\text{C} \quad T_w = 320^\circ\text{C} \quad T_f = 170^\circ\text{C}$$

$$\rho = 0.79 \text{ kg/m}^3 \quad \nu = 24.9 \times 10^{-6} \text{ m}^2/\text{s} \quad Pr = 0.681 \quad k = 0.017 \text{ W/mK}$$

$$Gr Pr = \frac{2.26 \times 10^{-3} \times 9.81 \times 300 \times 0.4^3}{(24.9 \times 10^{-6})^2} \times 0.681 = 0.46 \times 10^9 < 10^9 \text{ (Laminar)}$$

$$Nu_x = 0.6 (Gr Pr)^{0.2} = 32.5 \quad \bar{Nu}_L = \frac{4}{3} Nu_x = 43.33$$

$$h_L = \frac{Nu_L \cdot k}{L} = \frac{43.33 \times 0.017}{0.4} = 4 \text{ W/m}^2\text{K}$$



3. A vertical plate is maintained at  $40^\circ\text{C}$  in  $20^\circ\text{C}$  still air. Determine the height at which the boundary layer will turn turbulent if turbulence sets in at  $Gr \cdot Pr = 10^9$ . Repeat the problem for water flow at film temperature of  $30^\circ\text{C}$ . Determine the value of boundary layer thickness and average convection coefficient at the location where flow turns turbulent.

① Air:  $T_w = 40^\circ\text{C}$ ,  $T_\infty = 20^\circ\text{C}$  To find:  $\delta_x, L_c$

Properties of air  $T_f = \frac{40 + 20}{2} = 30^\circ\text{C}$

$\nu = 16 \times 10^{-6} \text{ m}^2/\text{s}$ ;  $\rho = 1.165 \text{ kg/m}^3$ ;  $Pr = 0.701$ ,

$$\beta = \frac{1}{30 + 273} = \frac{1}{303} = 3.3 \times 10^{-3} / ^\circ\text{C}$$

The flow becomes turbulent @  $GrPr = 10^9$ .

$$GrPr = \frac{\rho g \Delta T L_c^3}{\nu^2} \times Pr$$

$$10^9 = \frac{3.3 \times 10^{-3} \times 9.81 \times 20 \times L_c^3}{(16 \times 10^{-6})^2} \times 0.701$$

$$L_c = 0.82 \text{ m.}$$

$$\delta_x = 1.91 L_c^{0.5} Pr^{0.25} (0.701 + Pr)^{-0.25} \cdot Gr_c^{-0.25}$$

$$\delta_{x=L_c} = 0.022 \text{ m.} \rightarrow 22 \text{ mm.}$$

② Water:  $T_f = 30^\circ\text{C}$  Properties @  $30^\circ\text{C}$

$\rho = 998 \text{ kg/m}^3$   $\nu = 0.832 \times 10^{-6} \text{ m}^2/\text{s}$ .  $Pr = 5.68$ .

$$GrPr = \frac{\rho g \Delta T L_c^3}{\nu^2} \times Pr$$

$$10^9 = \frac{3.3 \times 10^{-3} \times 9.81 \times 20 \times L_c^3}{(0.832 \times 10^{-6})^2} \times 5.68$$

$$L_c = 0.057 \text{ m.} \rightarrow \delta_{x=L_c} = 1.38 \text{ mm.}$$

Comment: Critical length reduces for water. So it turns turbulent much faster for water.



19. Consider a surface 0.8 m high, kept at an angle of  $55^\circ$  from the horizontal at a constant wall temperature of  $40^\circ\text{C}$  in air at  $20^\circ\text{C}$ . Determine the value of convection coefficient and compare the same with that of similar vertical plate.

Sol:  $L = 0.8\text{m}$ ;  $\phi = 55^\circ$ ;  $T_w = 40^\circ\text{C}$   $T_\infty = 20^\circ\text{C}$   
To find ①  $h = ?$  ② Compare with vertical plate.

$$T_f = \frac{T_w + T_\infty}{2} = \frac{40 + 20}{2} = 30^\circ\text{C}$$

Properties @  $30^\circ\text{C}$

$$\rho = 1.165 \text{ kg/m}^3; \nu = 16 \times 10^{-6} \text{ m}^2/\text{s}; Pr = 0.701;$$

$$k = 0.02675 \text{ W/m}\cdot\text{K}$$

$$Gr = \frac{\beta \cdot g \cdot A \cdot T L^3}{\nu^2}$$

$$Gr = \frac{3.3 \times 10^{-3} \times 9.81 \times (40 - 20) \times 0.8^3}{(16 \times 10^{-6})^2} = 1.29 \times 10^9$$

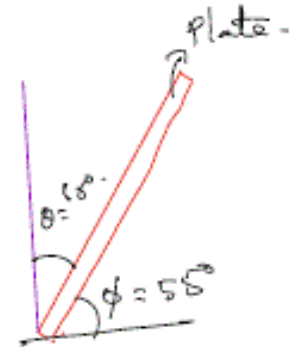
$$Ra = Gr \cdot Pr = 1.29 \times 10^9 \times 0.701 = 9.07 \times 10^8 < 10^9 \text{ (Laminar)}$$

Case a: Vertical

$$Nu = \left[ 0.825 + \frac{0.157(Ra)^{0.167}}{\left[ 1 + \frac{0.492}{Pr} \right]^{0.252}} \right]^2$$

$$Nu = 120.2$$

$$h = \frac{Nu \cdot k}{L} = \frac{120.2 \times 0.02675}{0.8} = 4.1 \text{ W/m}^2\cdot\text{K}$$



$$f = \frac{1}{20 + 27.3} = 3.3 \times 10^{-3} / \text{K}$$



Case b: Inclined  $\theta = 35^\circ$

$$Nu = \left[ 0.825 + \frac{0.187 (Ra \cos \theta)}{\left[ 1 + \frac{0.492}{Pr} \right]^{0.25} \left[ 1 + \frac{0.492}{Pr} \right]^{0.25}} \right]^2$$

$$Nu = 104.8$$

$$h = \frac{Nu k}{L} = \frac{104.8 \times 0.02675}{0.8} = 3.5 \text{ W/m}^2\text{K}$$

Assignment: ①  $\theta = 1^\circ$  to  $60^\circ$  [in steps of  $1^\circ$ ]

② Calculate  $h'$

③  $h \sqrt{L} \theta$ . Compare with  $h$  for vertical.

④ For constant heat flux inclined plates

$$T_e = T_w - 0.25 (T_w - T_a) \quad \beta = \frac{1}{T_f}$$

Properties at  $T_e$ .

Case a: Heated surface up.

Case b: Heated surface down.